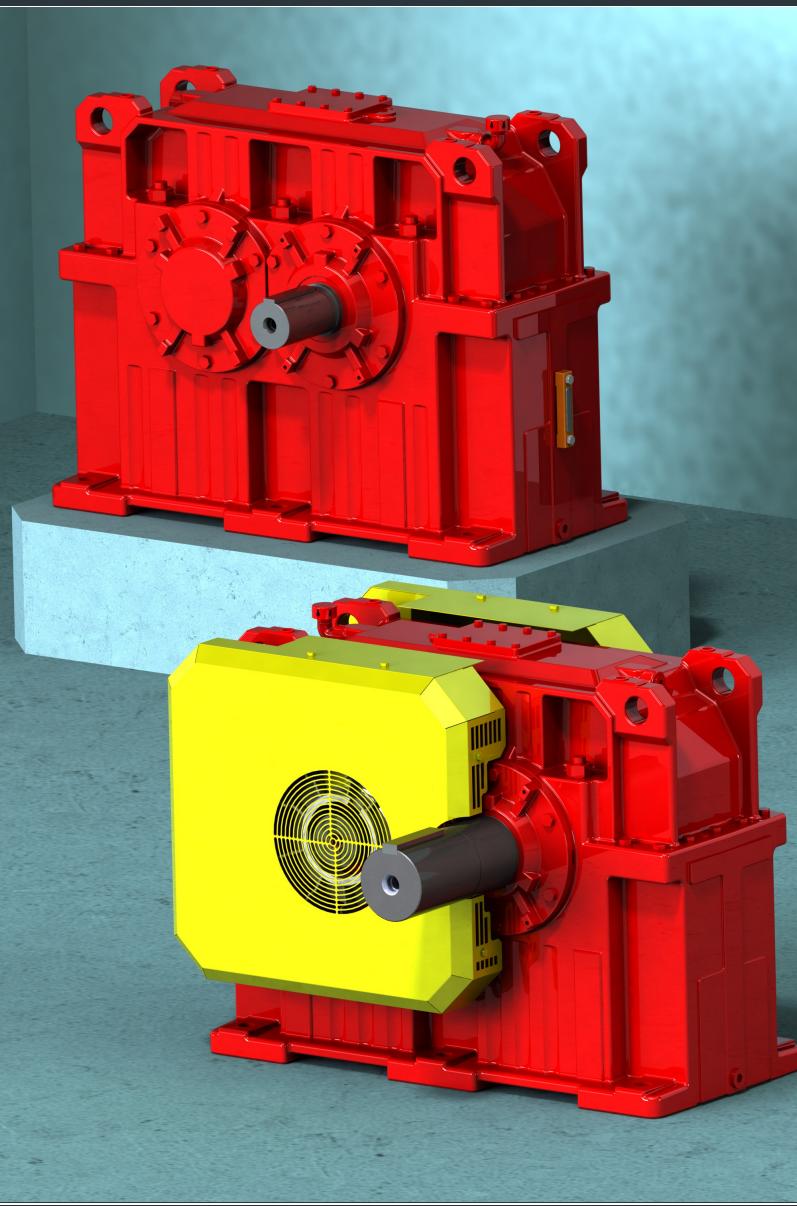


**SEW
EURODRIVE**

Catalog



**Industrial Gear Units
Single Stage Helical Gear Units
M1..N Series
Torque classes from 1.4 kNm - 189 kNm**





1 Introduction	4
1.1 The SEW-EURODRIVE Group of Companies.....	4
1.2 Products and systems from SEW-EURODRIVE.....	4
1.3 Copyright.....	5
1.4 Structure of the notes.....	5
1.5 Important information.....	5
2 Product Description and Overview of Types	6
2.1 Design features and advantages.....	6
2.2 Application areas.....	6
2.3 General information.....	6
2.4 Definition of icons used in dimension drawings.....	9
2.5 Shaft positions.....	9
2.6 Gear unit type designation.....	10
3 Project Planning for Drives	11
3.1 Project planning procedure	11
3.2 Project planning example: Sugar cane chipper drive.....	18
3.3 Lubricant selection/lubricant table.....	21
4 Technical Data	24
4.1 Nominal mechanical power ratings P_{N1}	24
4.2 Exact ratios i_{ex}	25
4.3 Nominal output torque M_{N2}	25
4.4 Thermal rating P_{TH}	26
5 Gear Unit Dimensions	30
5.1 M1PSF10 - 100N.....	30
5.2 M1PSF110 - 120N.....	32
6 Optional Accessories	34
6.1 Fan.....	34
6.2 Cooling coil system.....	35
6.3 Shaft end pump.....	35
6.4 Oil dipstick	36
6.5 Thermostatic water valve.....	36
6.6 Oil heater.....	37
6.7 Coupled Equipment - Sensors	38
Annex	41
A. Lubrication unit.....	41
B. Central lubrication system connections.....	44
C. Abbreviation key.....	45
D. Certificate.....	46



1. Introduction

1.1 The SEW-EURODRIVE group of companies

Global presence Driving the world with innovative drive solutions for all branches and every application. Products and systems from SEW-EURODRIVE are used in a multitude of applications worldwide. Be it in the automotive, building materials, food and beverage or metal-processing industry. The decision to use drive technology "made by SEW-EURODRIVE" stands for reliability for both functionality and investment.

We are represented in the most important branches of industry all over the world: with 13 manufacturing plants, 66 assembly plants in 47 countries and our comprehensive range of services, which we consider an integrative service that continues our commitment to outstanding quality.

Your ideal partner

Its global presence, extensive product range and broad spectrum of services make SEW-EURODRIVE the ideal partner for the machinery and plant construction industry when it comes to providing drive systems for demanding applications in all branches of industries and applications.

1.2 Products and systems from SEW-EURODRIVE

The products and systems from SEW-EURODRIVE are divided into the following 4 product groups:

1. Gearmotors and frequency inverters
2. Servo drive systems
3. Decentralized drive systems
4. Industrial gear units

The following table shows industrial gear units

Industrial gear units		
Helical and bevel-helical gear units <ul style="list-style-type: none"> • X series • MC series • ML series • ML..V..N series • M1..N series 	Helical and bevel-helical planetary gear units <ul style="list-style-type: none"> • P series (also as planetary gearmotors) • PMC series 	Drive packages <ul style="list-style-type: none"> • Application solutions with <ul style="list-style-type: none"> - Swing base - Gearmotors - Motors - Couplings - Drum and disk brakes - Lubrication systems for conveyor drives, bucket elevators, crushers, agitators, cooling towers, crane systems and much more

In addition to products and systems, SEW-EURODRIVE offers a comprehensive range of services. These include:

- Technical consulting
- Application software
- Seminars and training
- Extensive technical documentation
- International customer service

Visit our homepage at

www.sew-eurodrive.com

The website provides comprehensive information and services.



1.3 Copyright

© 2016 - SEW-EURODRIVE. All rights reserved.

Copyright law prohibits the unauthorized duplication, modification, distribution, and use of this document, in whole or in part.

1.4 Structure of the notes

The notes in this catalog are designed as follows:

Pictogram	Meaning
i	Useful information or tip.

1.5 Important information

Note the following points:



- The illustrations in the catalog are examples and are not binding.
- The specified fill quantities are non-binding guidelines. Use the marks on the oil dipstick or oil level glass to determine the oil level.
- The gear units are ready for operation when delivered, but are not filled with oil.
- Oil viscosity and grade must comply with those specified on the nameplate.
- The weights shown are non-binding average values.
- The buyer must provide protection against unintentional contact with moving parts. The applicable safety regulations of the country in which the unit will be used are to be followed.



2. Product Description and Overview of Types

2.1 Design features and advantages

- Independent industrial gear unit platform
- Helical gear units
- Extremely robust gear unit housing
- Reliable sealing and dust-proof systems
- Effective cooling systems
- Short delivery times for standard designs
- Worldwide service

2.2 Application areas

The M1..N series can be used in the following application areas:

- Pulp and paper industry
 - Pumps
 - Rubber and plastic industry
 - Food industry
- ...

2.3 General information

The subsequent chapters provide general information of the M1..N series gear units.

2.3.1 Nominal power, torques and input speeds

The nominal power and torques mentioned in the catalog depend on the input speed and are valid for a service factor of $F_S = 1.0$ and constant, unidirectional load. Only 70% of these values apply in the case of changing load directions.

The values in parameter table show nominal powers for input speeds of 1000 rpm to 1800 rpm, and nominal torques for input speeds of 1000 rpm. The nominal torque classes also apply to input speeds of 3% less than the synchronous speed.
Consult SEW-EURODRIVE for speeds >1800 rpm.

2.3.2 Housing

The gear unit housings are made of sturdy cast iron from M1PSF10N to M1PSF100N and steel from M1PSF110N to M1PSF120N.

The housing is a two-piece housing with horizontal parting line.

2.3.3 Gearing and shafts

The gearing with edge corrections is made of high-quality, tempered and ground case hardened steels. The output shafts are made of tough quenched and tempered steel.

2.3.4 Shaft bearing

Self-aligning roller bearings from well-known manufacturers are used.



2.3.5 Lubrication

Splash lubrication is used depending on the gear unit type and design. Bath and pressure lubrication can be used as option for the gear unit.

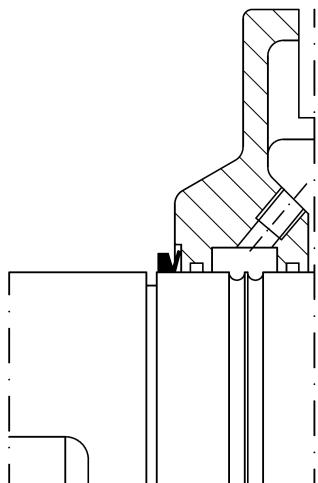
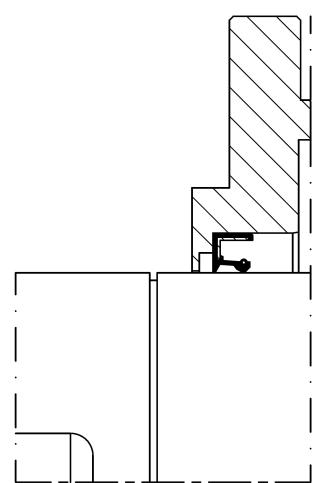
An oil level glass is used to visually check the oil level.

Instead of the oil drain plug, an oil drain valve can be provided. This ball valve allows for a drain line to be easily attached when changing the gear unit oil.

An oil heater might be necessary when ambient temperatures are low. The oil heater ensures that the oil is sufficiently liquid when starting up the gear unit in cold environments. SEW-EURODRIVE provides the oil heater with a preset thermostat that is attached to the gear unit.

2.3.6 Sealing system

Following sealing system can be used for input shaft and output shaft:

Labyrinth seal (M1PSF20N and above)	Single oil lip seal (M1PSF10N)
Labyrinth seal with V-Ring 	Single lip seal with dust protection lip 



The sealing system can be as option:

- Double lip seal with grease nipple
- Radial labyrinth seal with grease nipple

If need the sealing system above, please consult SEW-EURODRIVE.



Product Description and Overview of Types

General information

2.3.7 Modular accessories

Several optional accessories are available for industrial gear units.

- Fan
A cooling fan is used when the thermal rating of the gear unit is insufficient. Radial fan is bi-directional.
- Cooling coil system
Cooling coil system is used if the thermal rating of the gear unit is insufficient.
- Pressure lubrication with cooler
A pressure lubrication system with cooler is used when the thermal rating of the basic gear unit is not sufficient. A cooler is used in operating environments if a fan can not be used or is not sufficient.
- Shaft end pump
If condition is permitted, shaft end pump can be used, when adequate space must be allowed for it.
- Oil dipstick
Oil dipstick can be as optional equipment to check the oil level in gear housing.
- Oil heater
The oil heating system ensures that the oil is in a suitable viscosity range when starting up the gear unit in cold environments.
- Thermostatic water valve
The thermostatic water valve is used to regulate the water flow into cooler or cooling coil of the gear unit.
- Temperature sensor PT100
The PT100 temperature sensor can be used for measuring the oil or bearings temperature in the gear unit.
- SPM adapter (shock pulse adapter)
Adapters are installed on the gear unit housing for monitoring the vibration at various points on the gear unit.

2.3.8 Thermal rating

The thermal rating needs to be checked for every gear unit. The relevant values are listed in the selection tables on chapter 4.4.

2.3.9 Cooling

If the thermal rating of the gear unit is insufficient with natural cooling, you can choose one of the following cooling types:

- Fan (independent of direction of rotation) on the drive shaft.
- Water cooling coil in housing
- Oil supply system with oil-water heat exchanger or oil-air heat exchanger.

2.3.10 Surface and corrosion protection

On request, all gear units can be supplied with special surface protection for applications in extremely humid and chemically aggressive environments.



2.3.11 Weight data/oil quantities

Please note that all weights shown for the gear units in this catalog are guide values and do not include the lubricant. Refer to the dimension drawings for recommended oil quantities for the respective gear unit types. You find the exact gear unit weight in the order-specific dimension drawing.

2.3.12 International markets

SEW-EURODRIVE gear units conform to ISO and DIN specifications.



SEW-EURODRIVE is a member of the AGMA (American Gear Manufacturers' Association), if need gear units conform to AGMA specifications, please consult SEW-EURODRIVE.

2.3.13 Packaging

Standard packaging

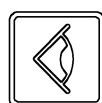
The gear unit is delivered on a pallet without cover.
Application: Land transport

Long-term packaging

The gear unit is delivered in a wooden box that is also appropriate for sea transport.
Application: Sea transport and /or long-term storage

2.4 Definition of icons used in dimension drawings

The following icons are possible to be used in dimension drawings:



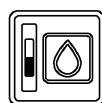
Visual inspection cover



Oil drainage



Oil filler



Oil level glass



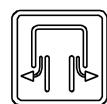
Oil sight glass



Oil stick



Lifting attachment



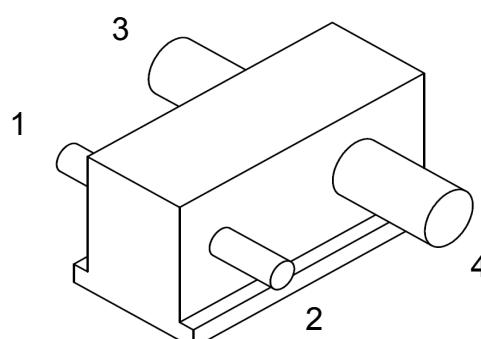
Breather plug



Oil heater

2.5 Shaft positions

The following shaft positions are possible for gear unit:



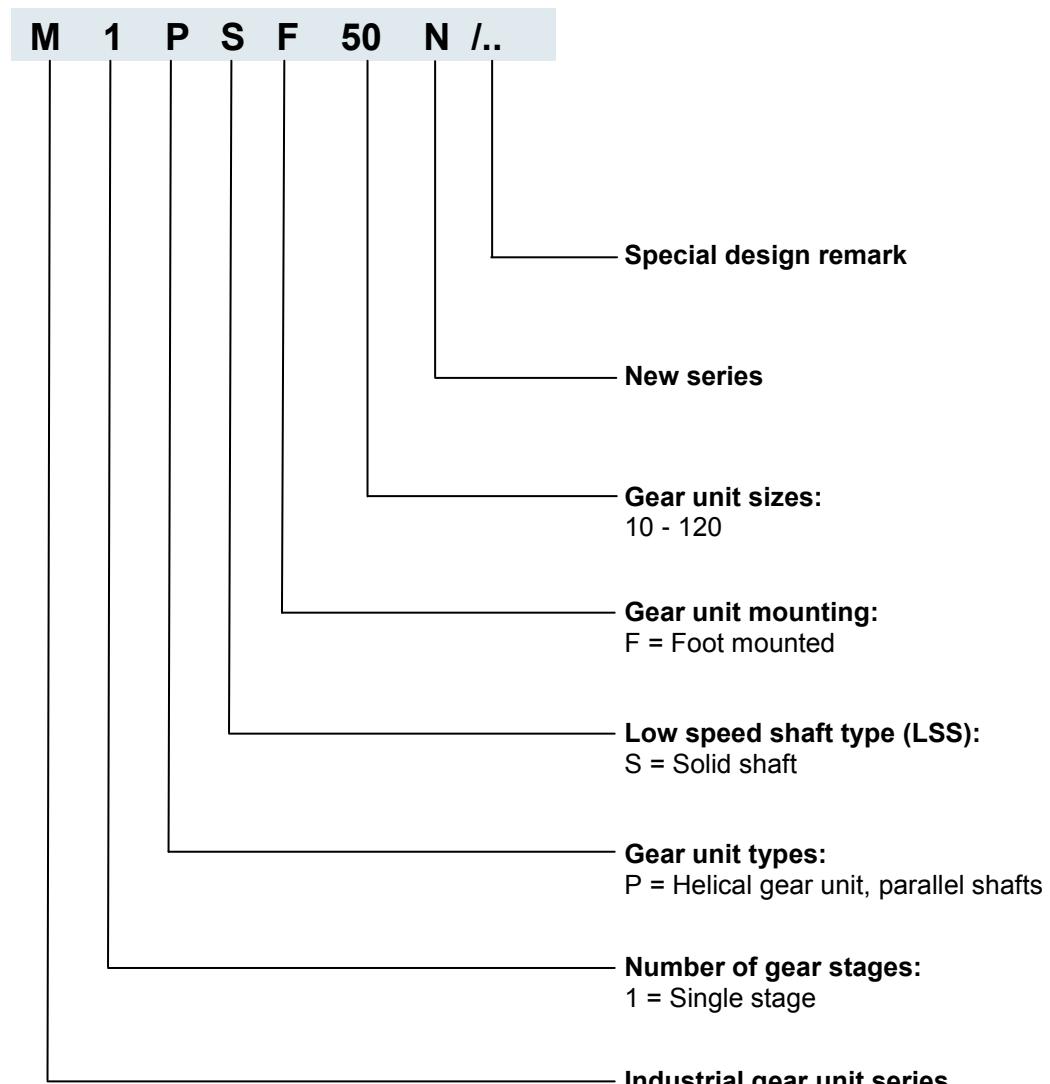


Product Description and Overview of Types

Gear unit type designation

2.6 Gear unit type designation

The designation of the gear unit is set up as follows:

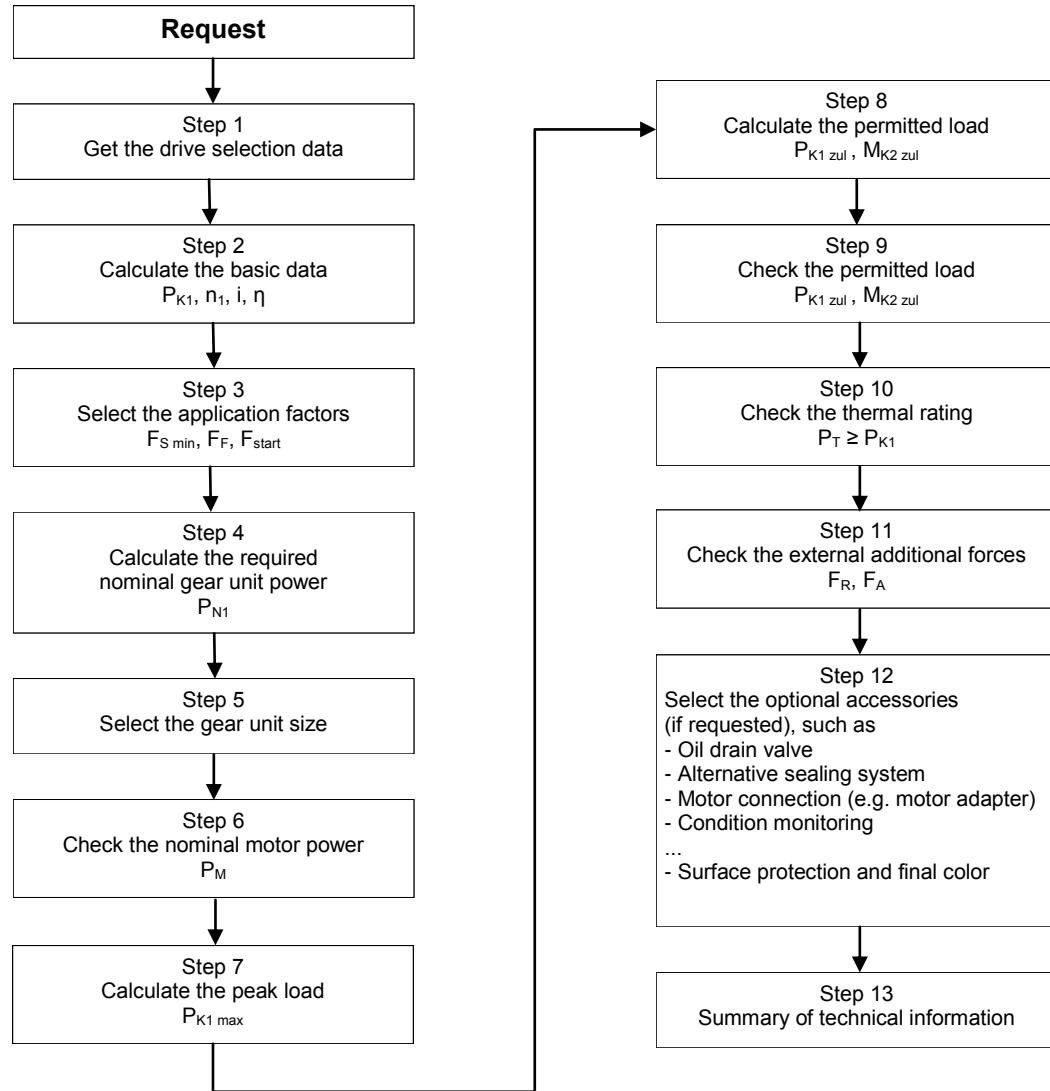




3. Project Planning for Drives

3.1 Project planning procedure

The following flow diagram illustrates the process for the project planning of M1..N series industrial gear units.



Step 1: Drive selection data

Get the drive selection data from customer:

- Machine on LSS (usually driven machine)
- Load characteristics
- Machine on HSS (usually driving machine)
- Gear unit requirement
 - Type and design
 - Service factor
 - Mounting and connection to machine on LSS
 - Loads on LSS and HSS



Project Planning for Drives

Project planning procedure

Step 2: Calculating the basic data – P_{K1} , n_1 , i , η

$$\text{Gear ratio } i = \frac{n_1}{n_2}$$

n_1 = Input speed (HSS) [rpm]
 n_2 = Output speed (LSS) [rpm]

$$\text{Operating power} \quad P_{K1} = \frac{M_{K2} \times n_2}{9.550 \times \eta} \quad [\text{kW}]$$

P_{K1} = Operating power on HSS [kW]
 M_{K2} = Output torque on LSS [kNm]
 n_2 = Output speed (LSS) [rpm]
 η = Efficiency

The following efficiency levels apply to gear units – η

The efficiency of the gear unit is mainly determined by the gearing and bearing friction as well by churning losses. The following **guide values** apply for M1..N series:

- M1PSF10 - 120N $\eta = 0.985$

Step 3: Selecting the application factors

Application service factor	$F_{S \min}$
Peak load factor	F_F
Startup factor	F_{start}

Application-specific service factor $F_{S \min}$

The application-specific service factor $F_{S \min}$ takes account of the typical load behavior with regard to the driven machine.

Recommended values with reference to

- Field of application
 - Type of driven machine
 - Operating period / day
- are given in the following table.



These tables apply only to gear units driven by electric motors. For other types of drive motors, the following correction values apply:

- Combustion engines with four or more cylinders:
 $F_{S \min}$ (selection table) + 0.25
- Combustion engines with one to three cylinders:
 $F_{S \min}$ (selection table) + 0.5



In the event of deviations from the typical load behavior, please consult SEW-EURODRIVE.



Field of application/ industry	Driven machine	Service factor operating period /day		
		< 3 h	3 - 10 h	> 10 h
Food industry	Crushers and mills	-	-	1.75
	Beet slicers	-	1.25	1.50
	Drying Drums	-	1.25	1.50
Pumps	Centrifugal pumps	1.15	1.35	1.45
	Reciprocating (single-cylinder)	1.35	1.50	1.80
	Reciprocating (multi-cylinder)	1.20	1.40	1.50
	Screw pump	-	1.25	1.50
	Rotary pumps (gear type, vane)	-	-	1.25
Pulp and paper industry	Debarking Drums and machines	1.55	1.8	-
	Rolls (pick-up, wire drive, wire suction)	-	1.8	2.00
	Drying cylinders (anti-friction bearings)	-	1.8	2.00
	Calenders (anti-friction bearings)	-	1.8	2.00
	Filters (pressure and vacuum)	-	1.8	2.00
	Beaters and chippers	1.55	1.75	2.00
	Jordan mills	-	1.50	1.75
	Presses (bark, felt, glue, suction)	-	-	1.75
	Reels	-	-	1.75
	Pulpers	1)	1)	1)
Rubber and plastic industry	Washer filters	-	-	1.50
	Yankee cylinders (dryers)	1)	1)	1)
	Extruders (plastic)	-	1.40	1.60
	Extruders (rubber)	-	1.50	1.80
	Rubber mills (two in a row)	1.55	1.75	2.00
	Rubber mills (three in a row)	-	1.50	1.75
	Warming mills	1.35	1.50	1.75
	Calenders	-	1.65	1.65
	Mills	1.55	1.75	2.00
	Mixing mills	1)	1)	1)



1) Consult SEW-EURODRIVE

Peak load factor F_F

The peak load factor F_F takes account of the overload capacity of the gearing and the rotating parts.

Peak load factor	Frequency of peak load per hour					
	1 ... 5	6 ... 20	21 ... 40	41 ... 80	81 ... 160	> 160
F_F	1.00	1.20	1.30	1.50	1.75	2.00



The gear units may only be overloaded for a short period of time. Individual peak loads must not last for more than ten seconds.



Project Planning for Drives

Project planning procedure

Startup factor F_{start}

The startup factor F_{start} takes account of the overload caused by startup.

Startup mode	Startup factor F_{start}
Direct	3.0
Soft start	1.8
Frequency inverter	1.5 ... 2.0 ¹⁾
Star / delta	1.3
Hydraulic coupling without delay chamber	2.0
Hydraulic coupling with delay chamber	1.6



1) Dependent on setting

Step 4: Calculating the required nominal gear unit power P_{N1}

Constant load direction – nominal power:

$$P_{N1} \geq P_{K1} \times F_{S \min} \quad [\text{kW}]$$

Reversing direction of load – nominal power:

$$P_{N1} \geq P_{K1} \times F_{S \min} \times 1.43 \quad [\text{kW}]$$

P_{N1} = Nominal gear unit power on HSS [kW]

P_{K1} = Operating power on HSS [kW]

$F_{S \min}$ = Application service factor

Step 5: Selecting the gear unit size

The selection of the gear unit size is based on the nominal gear unit power P_{N1} according to the tables showing the overview of speed and nominal gear ratio in section 4.



For input speeds $n_1 > 1800$ rpm, please consult SEW-EURODRIVE.

Step 6: Checking the nominal motor power P_M

$$P_M \geq P_{K1} = \frac{P_{K2}}{\eta} \quad [\text{kW}]$$

P_M = Nominal motor power [kW]

P_{K1} = Operating power on HSS [kW]

P_{K2} = Operating power on LSS [kW]

η = Efficiency

Step 7: Calculating the peak load $P_{K1 \max}$

Calculate the peak operating power $P_{K1 \max}$:

$$P_{K1 \max} = P_M \times F_{start} \quad [\text{kW}]$$

$P_{K1 \max}$ = Peak operating power on HSS [kW]

P_M = Nominal motor power [kW]

F_{start} = Startup factor



Step 8: Calculating the permitted peak load $P_{K1\ zul}$; $M_{K2\ zul}$

Permitted peak operating power $P_{K1\ zul}$:
Constant load direction

$$P_{K1\ zul} = \frac{2 \times P_{N1}}{F_F} \quad [\text{kW}]$$

Reversing load direction

$$P_{K1\ zul} = \frac{2 \times P_{N1}}{F_F} \times 0.7 \quad [\text{kW}]$$

$P_{K1\ zul}$ = Permitted peak operating power on HSS [kW]
 P_{N1} = Nominal gear unit power on HSS [kW]
 F_F = Peak load factor

Permitted peak output torque $M_{K2\ zul}$:
Constant load direction

$$M_{K2\ zul} = \frac{2 \times M_{N2}}{F_F} \quad [\text{kNm}]$$

Reversing load direction

$$M_{K2\ zul} = \frac{2 \times M_{N2}}{F_F} \times 0.7 \quad [\text{kNm}]$$

$M_{K2\ zul}$ = Permitted peak output torque [kNm]
 M_{N2} = Nominal gear unit torque [kNm]
 F_F = Peak load factor

3

Step 9: Checking the peak load $P_{K1\ zul}$; $M_{K2\ zul}$

The peak operating power $P_{K1\ max}$ must not exceed the permitted peak operating power $P_{K1\ zul}$.

Check the gear unit selection:

$$P_{K1\ max} \leq P_{K1\ zul} \quad [\text{kW}]$$

The peak output torque $M_{K2\ max}$ must not exceed the permitted peak output torque $M_{K2\ zul}$.

$$M_{K2\ max} \leq M_{K2\ zul} \quad [\text{kNm}]$$

$P_{K1\ max}$ = Peak operating power on HSS [kW]
 $P_{K1\ zul}$ = Permitted peak operating power on HSS [kW]
 $M_{K2\ max}$ = Peak output torque [kNm]
 $M_{K2\ zul}$ = Permitted peak output torque [kNm]



Project Planning for Drives

Project planning procedure

Step 10: Checking the thermal rating P_T

The thermal rating P_T of a gear unit is the power that a gear unit can transmit continuously without exceeding a certain oil temperature.

The thermal rating P_T depends on the following factors:

- Ambient temperature
- Air circulation and sunlight exposure at the installation site
- Installation altitude
- Heat conduction to the foundation at the installation site
- Gear unit type, size and gear ratio
- Type of gear unit external cooling
- Type of gear unit lubrication
- Lubricant type
- Cyclic duration factor



You can calculate the thermal rating P_T using the factors. The resulting calculation results are approximate values. Please consult SEW-EURODRIVE to determine the exact values.

$$P_T = P_{TH} \times f_1 \times f_4 \times f_L \times f_T \quad [\text{kW}]$$

P_T	= Thermal rating of the gear unit at existing ambient conditions [kW]
P_{TH}	= Thermal rating of the gear unit at define ambient conditions [kW]. See selection table on chapter 4.4.
f_1	= Altitude factor
f_4	= Operating cycle factor
f_L	= Lubrication factor
f_T	= Ambient temperature factor

Altitude factor f_1

The following table shows the altitude factor f_1

Altitude factor	Altitude H [m above sea level]				
	Up to 999	1000 ... 2000	2000 ... 3000	3000 ... 4000	4000 ... 5000
f_1	1.0	0.95	0.91	0.87	0.83

Operating cycle factor f_4

The following table shows the operating cycle factor f_4

Operating cycle factor	Time under load per hour in %				
	100	80	60	40	20
f_4	1.00	1.06	1.16	1.35	1.78

Lubrication factor f_L

The following table shows the lubrication factor f_L

Lubrication factor	Type of lubrication	
	Pressure	Bath or splash
f_L	1.10	1.00

Ambient temperature factor f_T

The following table shows the ambient temperature factor f_T

Ambient temperature factor	Ambient temperature [°C]				
	10	20	30	40	50
f_T	1.32	1.18	1.00	0.79	0.60



The gear unit's thermal rating must be at least as large as the operating power on the input shaft (HSS).

$$P_T \geq P_{K1} \quad [\text{kW}]$$

P_T = Thermal rating of the gear unit at existing ambient conditions [kW]
 P_{K1} = Operating power on HSS [kW]

Selecting the cooling system

If the thermal rating P_T of a gear unit is not sufficient, then you can use a cooling system with circulation cooling.

The proper size of the cooling system can be determined by approximation by means of the power loss P_L of the gear unit.

$$P_L = \left(P_{K1} - \frac{P_T}{2} \right) \times (1 - \eta) \quad [\text{kW}]$$

P_L = Power loss to be cooled [kW]
 P_{K1} = Operating power on HSS [kW]
 P_T = Thermal rating of the gear unit at existing ambient conditions [kW]
 η = Efficiency

The cooling requirements are determined by means of this power loss P_L . The required cooling system is selected by using the tables (see catalog of pressure lubrication unit) of the various circulation cooling types.

The power loss P_L of the gear unit must be smaller than the cooling capacity of the cooling system.

$$P_L < \text{cooling capacity of cooling system} \quad [\text{kW}]$$

In addition, the selection of an appropriate cooling system depends on the following factors:

- Actual power loss to be cooled
- Present cooling water temperature and volume flow
- Ambient temperature
- Ratio of oil quantity in the gear unit and oil volume flow of cooling system > 2



Please consult SEW-EURODRIVE for selecting the appropriate cooling system based on the ambient conditions of your system.

Step 11: Checking the external additional forces F_R, F_A :



If external additional forces load are present, please consult SEW-EURODRIVE.

Step 12: Selecting the optional equipment

Step 13: Summary of all the technical information



Project Planning for Drives

Project planning example: Sugar cane chipper drive

3.2 Project planning example: Sugar cane chipper drive

The following example shows the project planning for sugar cane chipper drive.

Technical data and application conditions of gear unit

- Foot-mounted helical gear unit with solid shaft (key connection)
- Output speed $n_2 = 740 \text{ rpm}$
- Output torque $M_{K2} = 15 \text{ kNm}$
- Peak output torque on output shaft $M_{K2 \max} = 30 \text{ kNm}$
- Input speed $n_1 = 1180 \text{ rpm}$
- Motor power $P_M = 1200 \text{ kW}$
- Operating time: on average 40% per hour, 12 hours per day
- The gear unit is to be started up once per hour and subjected to frequent shock load from the in-feed (frequency of maximum output torque approx. 100 per hour)
- The gear unit is to be used in open area under dusty environment, and an ambient temperature of 40°C
- Motor control via frequency inverter
- Installation at sea level.
- Customer requirement: Service factor ≥ 2.0

Step 1: Getting drive selection data

Step 2: Calculating the basic data

Calculate the gear unit reduction ratio i using the following formula:

$$i = \frac{n_1}{n_2} = \frac{1180 \text{ rpm}}{740 \text{ rpm}} = 1.595$$

i = Gear ratio
 n_1 = Input speed (HSS) [rpm]
 n_2 = Output speed (LSS) [rpm]

This value is used to specify the nominal gear ratio $i_N = 1.6$.

Operating power P_{K1} :

$$P_{K1} = \frac{M_{K2} \times n_2}{9.550 \times \eta} = \frac{15 \text{ kNm} \times 740 \text{ rpm}}{9.550 \times 0.985} = 1180 \text{ kW}$$

P_{K1} = Operating power on HSS [kW]
 M_{K2} = Operating torque on LSS [kNm]
 n_2 = Output speed LSS [rpm]
 η = Efficiency = 0.985 (See page 12)

Step 3: Selecting the application factors

Application-specific service factor (Chipper drive, $t > 10 \text{ h/day}$)	$F_{S \min} = 2.0$
Peak load factor (81...160 peak loads per hour)	$F_F = 1.75$
Startup factor (Frequency inverter)	$F_{start} = 2.0$



Step 4: Calculating the required nominal gear unit power

The required nominal gear unit power P_{N1} :

$$P_{N1} \geq P_{K1} \times F_{S\ min} = 1180 \text{ kW} \times 2.0 = 2360 \text{ kW}$$

P_{N1} = Nominal gear unit power on HSS [kW]
 P_{K1} = Operating power on HSS [kW]
 $F_{S\ min}$ = Application service factor

Step 5: Selecting the gear unit size

Select a gear unit of the next larger power class according to the nominal power P_{N1} tables showing in chapters 4.1.

- Nominal gear unit power $P_{N1} = 2798 \text{ kW}$, see selection table on page 24.
- Gear unit type **M1PSF60N**
- Nominal gear ratio $i_N = 1.6$, Exact gear ratio $i_{ex} = 1.579$, see selection table on page 25.
- Nominal gear unit torque $M_{N2} = \frac{P_{N1} \times 9.550 \times \eta}{n_2} = \frac{2798 \text{ kW} \times 9.550 \times 0.985}{1180 \text{ rpm} / 1.579} \approx 35.2 \text{ kNm}$
- Thermal rating $P_{TH} = 328 \text{ kW}$ without additional cooling, see selection table on page 27.

3

Step 6: Checking the nominal motor power P_M

$$P_M = 1200 > P_{K1} = 1180 \text{ kW}$$

P_M = Nominal motor power [kW]
 P_{K1} = Operating power on HSS [kW]

Step 7: Calculating the peak load $P_{K1\ max}$

$$P_{K1\ max} = P_M \times F_{start} = 1200 \text{ kW} \times 2 = 2400 \text{ kW}$$

$P_{K1\ max}$ = Peak operating power based on motor [kW]
 P_M = Nominal motor power [kW]
 F_{start} = Startup factor

Step 8: Calculating the permitted peak load $P_{K1\ zul}$ & $M_{K2\ zul}$

$$P_{K1\ zul} = \frac{2 \times P_{N1}}{F_F} = \frac{2 \times 2798 \text{ kW}}{1.75} = 3198 \text{ kW}$$

$P_{K1\ zul}$ = Permitted peak operating power [kW]
 P_{N1} = Nominal gear unit power [kW]
 F_F = Peak load factor

$$M_{K2\ zul} = \frac{2 \times M_{N2}}{F_F} = \frac{2 \times 35.2 \text{ kNm}}{1.75} = 40.2 \text{ kNm}$$

M_{N2} = Nominal gear unit torque [kNm]
 $M_{K2\ zul}$ = Permitted peak output torque [kNm]
 F_F = Peak load factor



Project Planning for Drives

Project planning example: Sugar cane chipper drive

Step 9: Checking the permitted peak load $P_{K1\ zul}$ & $M_{K2\ zul}$

The peak operating power $P_{K1\ max}$ must not exceed the permitted peak operating power $P_{K1\ zul}$.

$$P_{K1\ max} = 2400 \text{ kW} < P_{K1\ zul} = 3198 \text{ kW}$$

The peak output torque $M_{K2\ max}$ must not exceed the permitted peak output torque $M_{K2\ zul}$.

$$M_{K2\ max} = 30 \text{ kNm} < M_{K2\ zul} = 40.2 \text{ kNm}$$

$P_{K1\ max}$ = Peak operating power based on motor [kW]

$P_{K1\ zul}$ = Permitted peak operating power [kW]

$M_{K2\ max}$ = Peak output torque [kNm]

$M_{K2\ zul}$ = Permitted peak output torque [kNm]

→ This means you can use the selected gear unit size.

Step 10: Checking the thermal rating

$$P_T = P_{TH} \times f_1 \times f_4 \times f_L \times f_T = 328 \text{ kW} \times 1.0 \times 1.35 \times 1.1 \times 0.79 = 384.8 \text{ kW}$$

P_T = Thermal rating of the gear unit at existing ambient conditions [kW]

P_{TH} = Thermal rating of the gear unit at define ambient conditions [kW]

f_1 = Altitude factor (see page 16)

f_4 = Operating cycle factor (see page 16)

f_L = Lubrication factor (see page 16)

f_T = Ambient temperature factor (see page 16)

The operating power P_{K1} must not exceed the thermal rating P_T ($P_{K1} \leq P_T$). Additional cooling is required if $P_{K1} > P_T$

$$1180 \text{ kW} > 384.8 \text{ kW}$$

→ Thermal rating is not sufficient at 40 °C without additional cooling.

→ Oil/water cooler or oil/air cooler is needed.



Please consult SEW-EURODRIVE for selecting the appropriate pressure lubrication with cooler based on the ambient conditions of your system.

Step 11: Checking the external additional forces

There are no external additional forces.

Step 12: Selecting the optional equipment

- Standard labyrinth seal on the input and output shafts.
- Pressure lubrication with cooler and internal piping

Step 13: Summary of all the technical information

- Gear unit type M1PSF60N
- Gear ratio $i_{ex} = 1.579$
- Final output speed: 747 rpm
- Nominal power $P_{N1} = 2798 \text{ kW}$
- Motor $P_M = 1200 \text{ kW}$
- Final $F_S = P_{N1}/P_{K1} = 2798 \text{ kW}/1180 \text{ kW} = 2.37$
- Standard labyrinth seal the input and output shafts
- Pressure lubrication with cooler and internal piping



3.3 Lubricant selection/lubricant table



The oil viscosity and type (mineral/synthetic) to be used are determined by SEW-EURODRIVE specifically for each order. This information is noted in the order confirmation and on the gear unit's nameplate. Any deviation from these conditions requires consultation with SEW-EURODRIVE.

3.3.1 General information on selecting the oil

Unless a special arrangement is made, SEW-EURODRIVE delivers the drives without oil fill. This does not apply to auxiliary drives and primary gear units.



This means the gear unit must be filled with the correct oil grade and quantity before startup. You find the corresponding information on the nameplate of the gear unit.

The following tables provide an overview of mineral and synthetic oils.

Mineral oil Standard

Lubrication oils are divided into ISO VG viscosity classes according to ISO 3448 and DIN 51519.

ISO class	ISO 6743-6 designation	DIN 51517-3 designation	AGMA 9005-D94 designation
220	ISO-L-CKC 220	DIN 51517-CLP 220	AGMA 5 EP
320	ISO-L-CKC 320	DIN 51517-CLP 320	AGMA 6 EP
460	ISO-L-CKC 460	DIN 51517-CLP 460	AGMA 7 EP

Synthetic oil Standard

Lubrication oils are divided into ISO VG viscosity classes according to ISO 3448 and DIN 51519.

ISO class	ISO 6743-6 designation	DIN 51519 designation	AGMA 9005-D94 designation
220	ISO-L-CKT 220	CLP HC 220	AGMA 5 EP
320	ISO-L-CKT 320	CLP HC 320	AGMA 6 EP
460	ISO-L-CKT 460	CLP HC 460	AGMA 7 EP

In addition to the required viscosity, the oil must fulfill the following criteria:

- CLP oils according to DIN 51517-3
- Micro-pitting test according to FVA, FV no. 54/ I-IV, GFT class high, damage force level >10

If synthetic oil is used, SEW-EURODRIVE recommends polyalphaolefin-based oil (CLP HC).



Project Planning for Drives

Lubricant selection/lubricant table

3.3.2 Permitted lubricants



- The standard for viscosity and oil grade is the type of oil that is specified by SEW-EURODRIVE in the order (see order confirmation and nameplate).
- Please consult SEW-EURODRIVE if you use bio and food grade lubricants or polyglycol oils.
- Check the compatibility of the greases and oils used.

Key to the lubricant table

The lubricant table shows the permitted lubricants for SEW-EURODRIVE gear units. Please note the abbreviations, meaning of shading and notes.

CLP = Mineral oil
 CLP HC = Synthetic polyalphaolefin
 E = Ester oil (water hazard classification 1)

 = Mineral lubricant
 = Synthetic lubricant

- 3) = Use lubricants only when service factor $F_S \geq 1.3$
 4) = Take into account critical startup behavior at low ambient temperatures
 6) = Ambient temperature

 = Lubricant for the food industry (food grade oil)

 = Biodegradable oil (lubricant for agriculture, forestry, and water management)

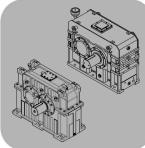
Project Planning for Drives

Lubricant selection/lubricant table

Lubricant table

470490305

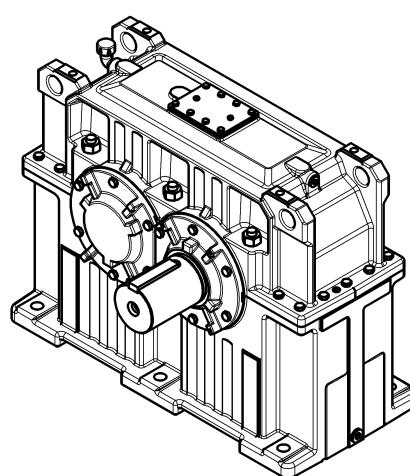
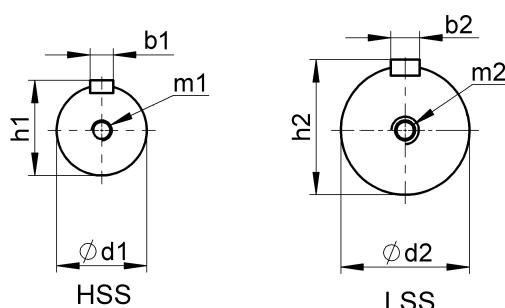
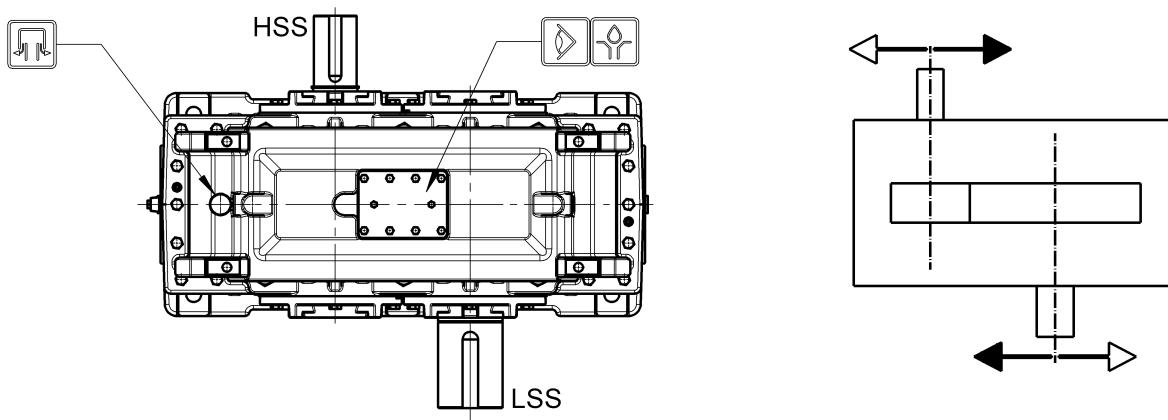
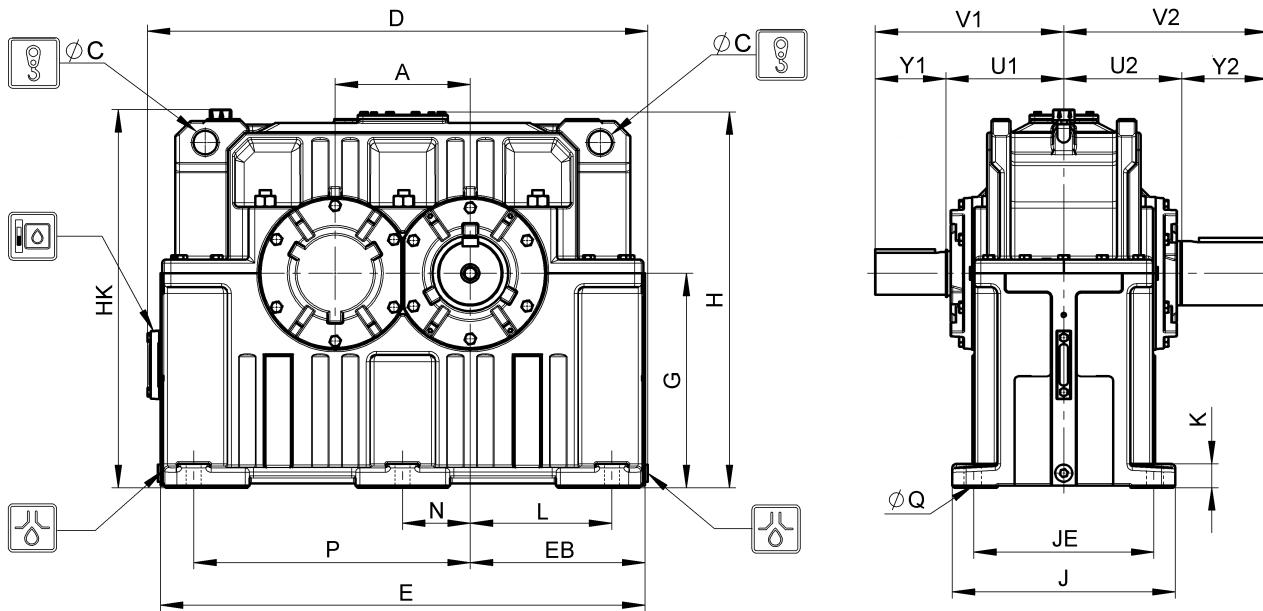
6)			ISO,NLG DIN (ISO)		bp						Tribol		Q8	
-50	0	+50	+100	Mobilgear XMP 320 Mobilgear 600XP 320	Shell Omala F 320	BP Energol GR-XP-320	Klüberööl GEM 1-320 N	Aral Degol BG 320	Meropa 320	Alpha SP 320 Tribol 1100/320	Optigear BM 320	Renolin CLP 320 Plus Renolin High Gear 320	Goya NT 320	Carter EP 320
-10		Standard	+40	CLP CC	VG 320									
-20	+20	CLP CC	VG 150 ³⁾	Mobilgear XMP 150 Mobilgear 600XP 150	Shell Omala F 220	BP Energol GR-XP-150	Klüberööl GEM 1-150 N	Aral Degol BG 150	Meropa 150	Alpha SP 150 Tribol 1100/150	Optigear BM 150	Renolin CLP 150 Plus Renolin High Gear 150	Goya NT 150	
-15	+30	CLPCC	VG 220	Mobilgear XMP 220 Mobilgear 600XP 220	Shell Omala F 220	BP Energol GR-XP-220	Klüberööl GEM 1-220 N	Aral Degol BG 220	Meropa 220	Alpha SP 220 Tribol 1100/220	Optigear BM 220	Renolin CLP 220 Plus Renolin High Gear 220	Goya NT 3220	Carter EP 220
-5	+45	CLP CC	VG 460	Mobilgear XMP 460 Mobilgear 600XP 460	Shell Omala F 460	BP Energol GR-XP-460	Klüberööl GEM 1-460 N	Aral Degol BG 460	Meropa 460	Alpha SP 460 Tribol 1100/460	Optigear BM 460	Renolin CLP 460 Plus Renolin High Gear 460	Goya NT 460	Carter EP 460
0	+50	CLP CC	VG 680	Mobilgear XMP 680 Mobilgear 600XP 680	Shell Omala F 680	BP Energol GR-XP-680	Klüberööl GEM 1-680 N	Aral Degol BG 680	Meropa 680	Alpha SP 680 Tribol 1100/680	Optigear BM 680	Renolin CLP 680 Plus Renolin High Gear 680	Goya NT 680	Carter EP 680
Standard	+40	CLP HC	VG 320	Mobilgear SHC XMP 320 Mobil SHC 632	Shell Omala HD 320	BP Energyn EP-XF-320	Klübersynth GEM 4-320 N	Pinnacle EP 320		Alphasyn EP 320	Optigear Synthetic X 320	Renolin CLP 320 Plus Renolin High Gear Synth 320	El Greco 320	Carter SH 320
4)	+10	CLP HC	VG 68 ³⁾	Mobil SHC 626 XMP 150	Shell Omala HD 68	BP Energyn EP-XF-68	Klübersynth GEM 4-68 N	Pinnacle EP 150		Alphasyn EP 150	Optigear Synthetic X 68	Renolin CLP 68		
4)	+20	CLP HC	VG 150	Mobilgear SHC XMP 150 Mobil SHC 630	Shell Omala HD 150	BP Energyn EP-XF-150	Klübersynth GEM 4-150 N	Pinnacle EP 150		Alphasyn EP 150	Optigear Synthetic X 150	Reolin Unisyn CLP 150	El Greco 150	Carter SH 150
4)	+35	CLP HC	VG 220	Mobilgear SHC XMP 220 Mobil SHC 630	Shell Omala HD 220	BP Energyn EP-XF-220	Klübersynth GEM 4-220 N	Pinnacle EP 220		Alphasyn EP 220	Optigear Synthetic X 220	Renolin CLP 220 Plus Renolin High Gear Synth 220	El Greco 220	Carter SH 220
-30	+30	CLP HC	VG 460	Mobilgear SHC XMP 460 Mobil SHC 634	Shell Omala HD 460	BP Energyn EP-XF-460	Klübersynth GEM 4-460 N	Pinnacle EP 460		Alphasyn EP 460	Optigear Synthetic X 460	Renolin CLP 460 Plus Renolin High Gear Synth 460	El Greco 460	Carter SH 460
-10	+60	CLP HC	VG 680	Mobilgear SHC XMP 680 Mobil SHC 636	Shell Omala HD 680	BP Energyn EP-XF-680	Klübersynth GEM 4-680 N	Pinnacle EP 680		Alphasyn EP 680	Optigear Synthetic X 680	Reolin Unisyn CLP 680	El Greco 680	Carter SH 680
-10	+30	CLP HC NSF H1	VG 68 ³⁾		Shell Cassida Fluid GL 68									
-20	+20		VG 220 ³⁾		Shell Cassida Fluid GL 220									
4)	+10		VG 460 ³⁾		Shell Cassida Fluid GL 460									
-20	+40	E	VG 460		Klüberbio CA2-460								Plantogear 460 S	



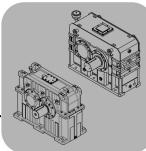
Gear Unit Dimensions M1PSF10 - 100N

5. Gear Unit Dimensions

5.1 M1PSF10 - 100N

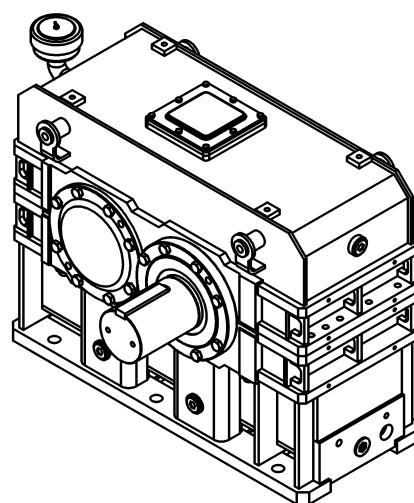
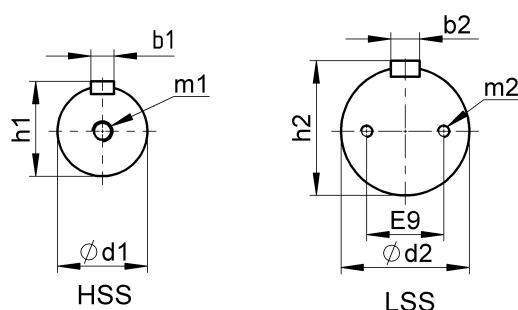
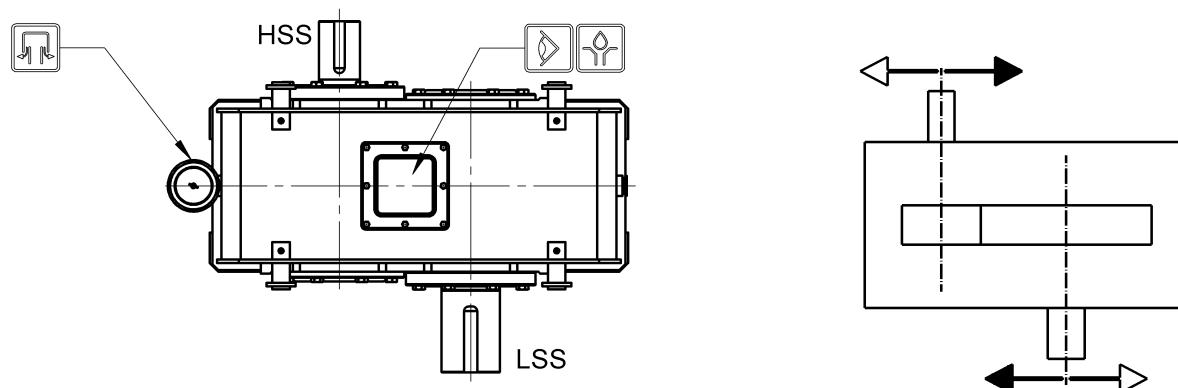
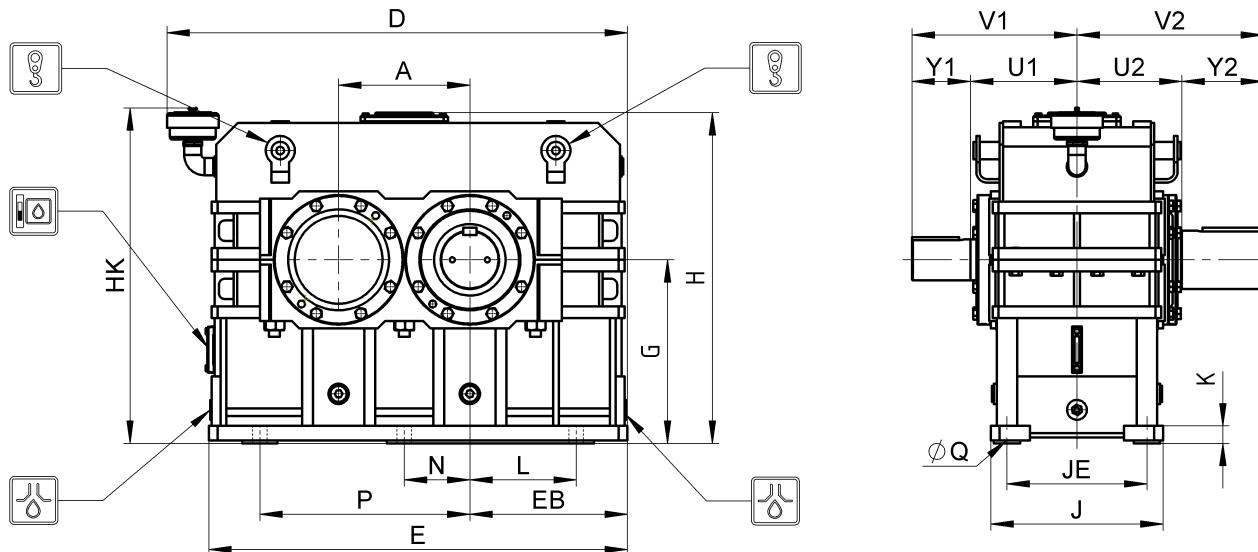


- Keys: ISO773/2491, DIN6885.
- Material of gear housing is cast iron.
- Oil drain valve can be as option.

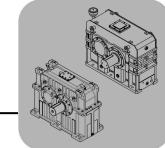


Gear Unit Dimensions M1PSF110 - 120N

5.2 M1PSF110 - 120N



- Keys: ISO773/2491, DIN6885.
- Material of gear housing is steel.
- Oil drain valve can be as option.



5.2 M1PSF110 - 120N

Size	Housing Dimensions [mm]								Foot Mounting [mm]					
	A	D	E	EB	G	H	HK	J	JE	K	L	N	P	Q
110	500	1724	1580	590	710	1270	1288	660	540	70	405	250	805	42
120	560	1904	1760	650	760	1370	1388	768	600	80	450	280	890	42

Size	i _N	HSS Dimensions [mm]							LSS Dimensions [mm]							Weight [kg]	Oil Quantity [l]	
		U1	Y1	V1	d1	b1	h1	m1	U2	Y2	V2	d2	b2	h2	m2	E9		
110	1.0 ≤ i ≤ 3.3	400	240	640	180m6	45h9	190	M30x60	400	280	680	220m6	50h9	231	2-M20x36	132	3760	185 ³⁾
	3.3 < i ≤ 4.8		200	600	150m6	36h9	158	M30x60										
	4.8 < i < 7.1		165	565	120m6	32h9	127	M24x50										
120	1.0 ≤ i ≤ 3.3	463	280	743	200m6	45h9	210	+)	468	330	798	260m6	56h9	272	2-M24x45	156	5630	185 ³⁾
	3.3 < i ≤ 4.5		240	703	180m6	45h9	190	M30x60										
	4.5 < i < 7.1		200	663	150m6	36h9	158	M30x60										

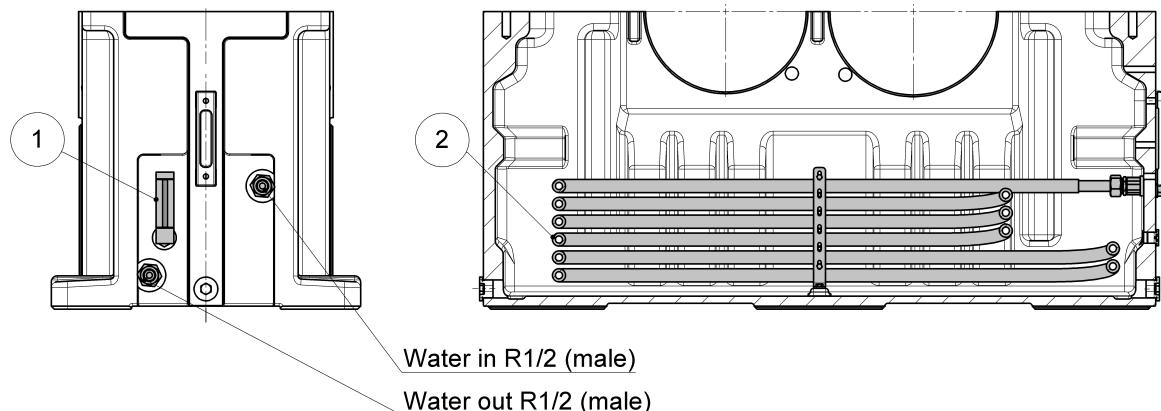


- The weight data listed above do not include the lubricant.
- The weight and oil quantity data listed above are guide values.
- 3) Pressure lubrication is used for the gear unit.
- + Please consult SEW-EURODRIVE.



6.2 Cooling coil system

Cooling coil is used with bath and splash lubrication if the thermal rating of the gear unit is insufficient..



Components:

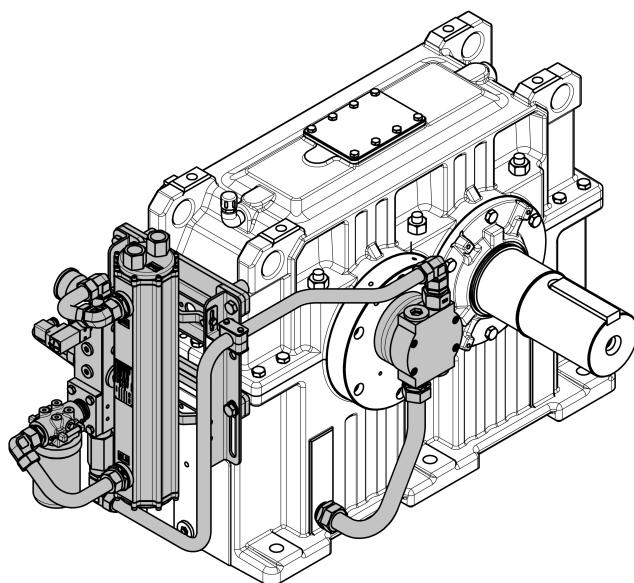
1. Thermometer scale 0 ... 100°C (+32 ... +212°F)
2. Cooling pipe



- Cooling coil is not available for M1PSF10N, M1PSF100N and above.
- Thermostatic water valve can be as option.

6.3 Shaft end pump

If condition is permitted, shaft end pump can be used from M1PSF20N to M1PSF120N, when adequate space must be allowed for it. Available pump outputs Q_p : 18, 28, 44, 65 dm³/min at 1500 rpm.



- The shaft end pump is used when pressure lubrication is needed and a motor pump is not desirable.
- A minimum input speed is required for the shaft end pump to operate properly.
- Gear unit continuous running is required if using shaft end pump.
- If using variable input speeds (e.g. inverter-controlled drives) or if intending to change the input speed of gear unit equipped with a shaft end pump, please be sure to consult SEW-EURODRIVE.

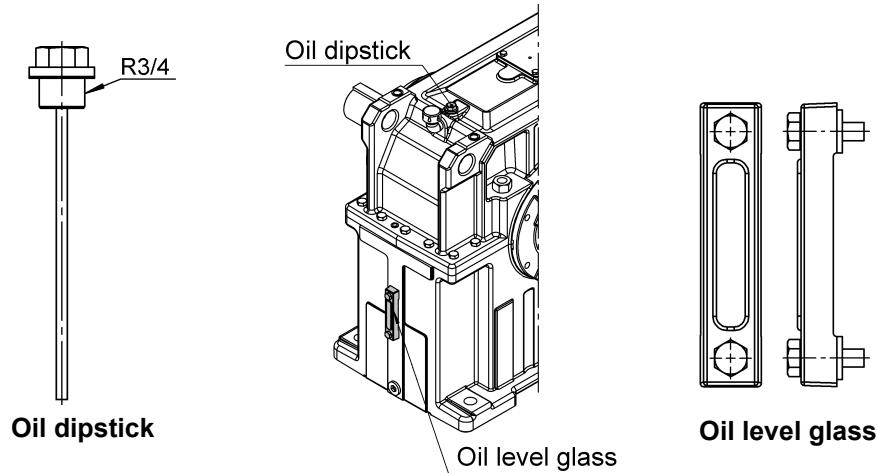


Optional Accessories

Oil dipstick / Thermostatic water valve

6.4 Oil dipstick

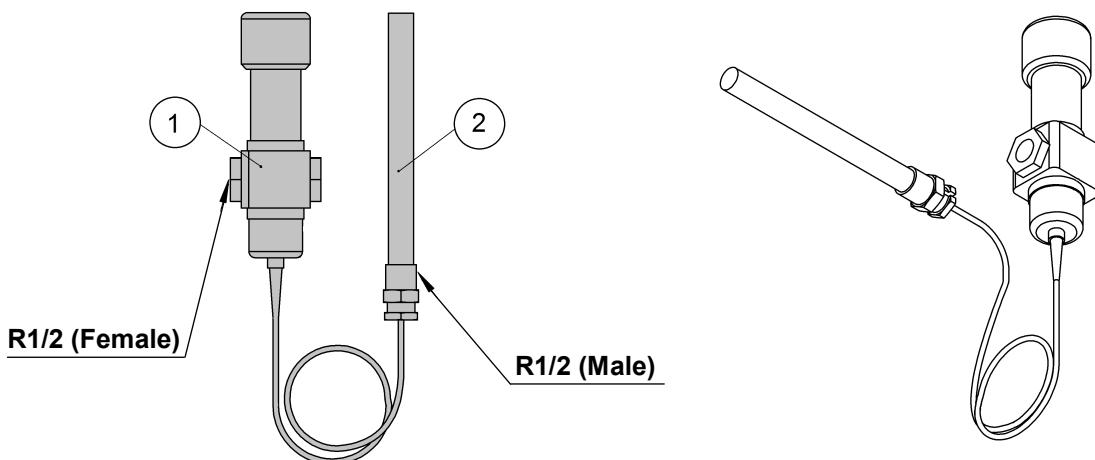
Oil dipstick can be as optional equipment to check the oil level.
Oil level glass is as an standard equipment for M1..N series gear units.



- Oil level can be checked only when the gear unit is not running.
- Water and dirt penetration into the gear unit must be prevented while checking the oil level.

6.5 Thermostatic water valve

The thermostatic water valve is used to regulate the water flow into cooler (lubrication unit) or cooling coil of the gear unit and so adjust the amount of cooling.



Components:

1. Thermostatic valve
2. Sensor

Technical data:

- | | |
|---------------------------------|-------------|
| • Maximum working pressure*) | 10 bar |
| • Minimum working pressure *) | 2 bar |
| • Maximum differential pressure | 7 bar |
| • Maximum water temperature *) | 25°C (40°C) |



*) when using the water valve for lubrication unit or cooling coil.



6.6 Oil heater

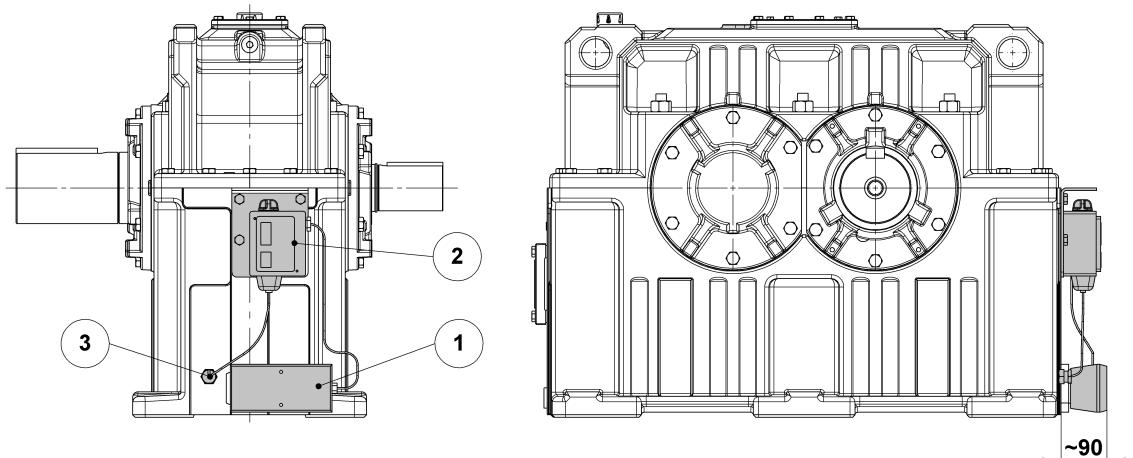
An oil heater is required to ensure lubrication during a cold gear unit startup when the ambient temperature is low.

The minimum starting temperature in splash and bath lubricated gear:

- ISO VG220 (Mineral oil): Minimum starting temperature -16°C
- ISO VG320 (Mineral oil): Minimum starting temperature -12°C
- ISO VG460 (Mineral oil): Minimum starting temperature -8°C

In pressure lubricated gear units the oil viscosity in minimum starting temperature must be lower than 2000 cSt.

- ISO VG220 (Mineral oil): Minimum starting temperature 10°C
- ISO VG320 (Mineral oil): Minimum starting temperature 15°C
- ISO VG460 (Mineral oil): Minimum starting temperature 20°C



Components:

1. Oil heater
2. Thermometer
3. Sensor

M1..N	Frequency 50Hz		Frequency 60Hz	
	Power [W]	Voltage [V]	Power [W]	Voltage [V]
10	-	-	-	-
20	330	230	330	230
30	670	230	670	230
40	670	230	670	230
50	1000	400Y/230Δ	1000	400Y/230Δ
60	1000	400Y/230Δ	1000	400Y/230Δ
70	1500	400Y/230Δ	1500	400Y/230Δ
80	1500	400Y/230Δ	1500	400Y/230Δ
90	2000	400Y/230Δ	2000	400Y/230Δ
100	2000	400Y/230Δ	2000	400Y/230Δ
110	2330	400Y/230Δ	2330	400Y/230Δ
120	3000	400Y/230Δ	3000	400Y/230Δ



Optional Accessories

Coupled equipment - Sensor

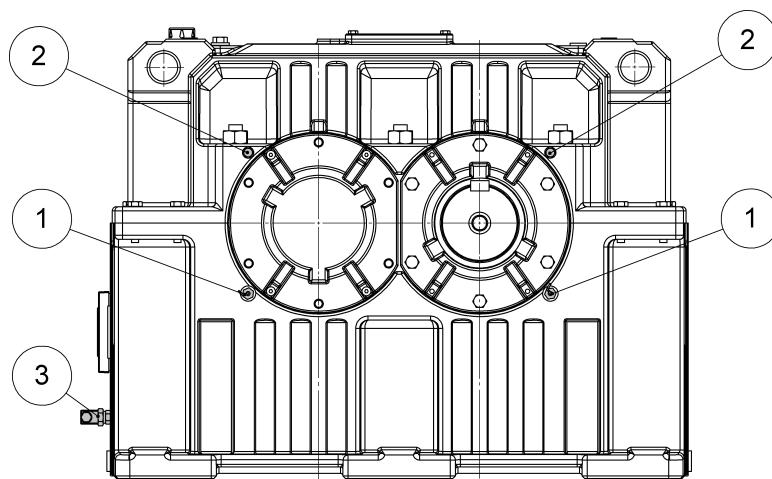
6.7 Coupled equipment - Sensors

6.7.1 Shock impulse sensor

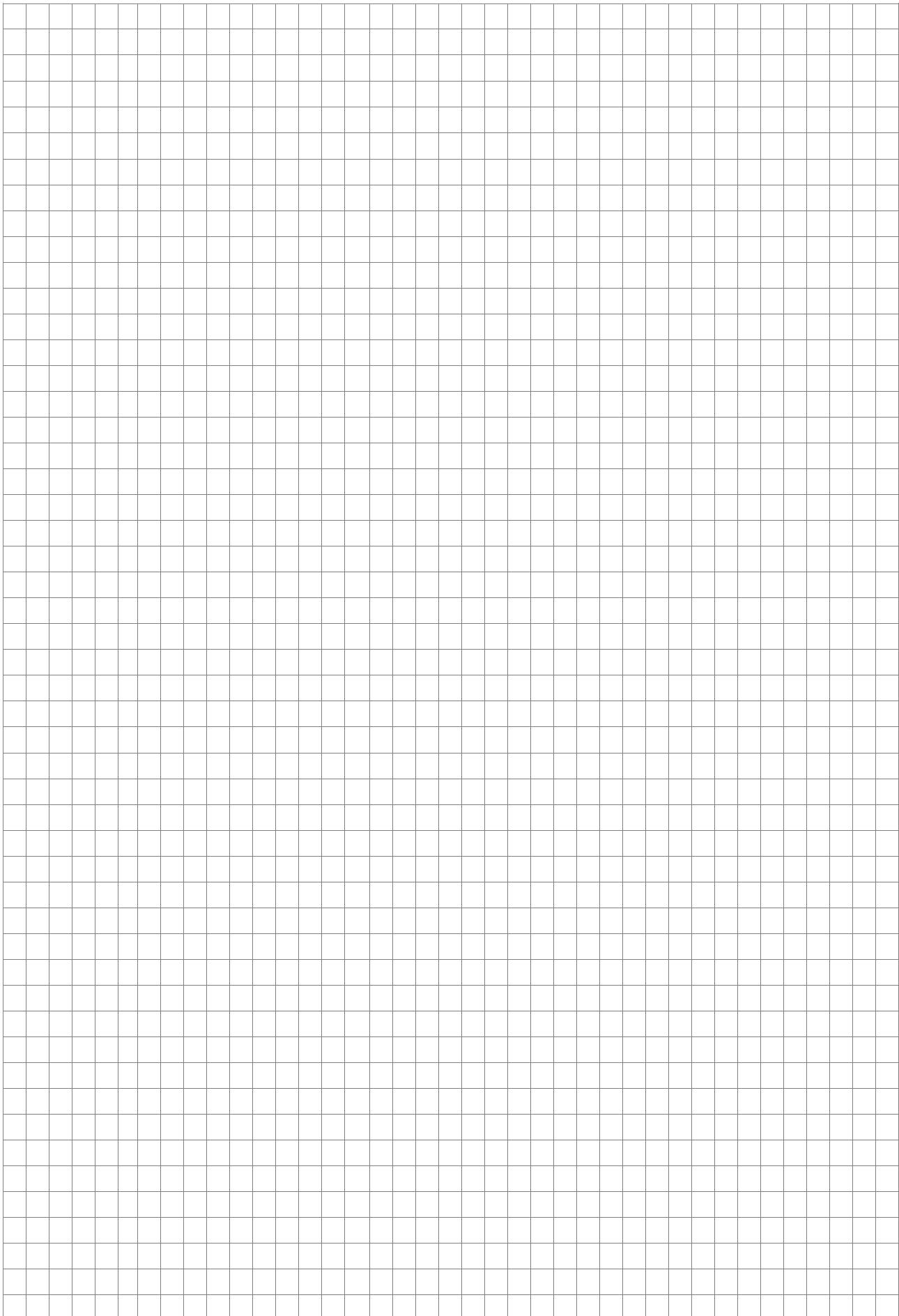
SIS1	SIS2	SIS3								
Nipple STD-2 UNC5/16 and cover STE-01	Nipple 32000 and cover 81025	Shock pulse transducer 40000 and TNC cable connector 13008								
	<table border="1"> <tr> <th>g</th> <th>L [mm]</th> </tr> <tr> <td>M8 (UNC 5/16")</td> <td>24, 113, 202, 291</td> </tr> </table>	g	L [mm]	M8 (UNC 5/16")	24, 113, 202, 291	<table border="1"> <tr> <th>g</th> <th>L [mm]</th> </tr> <tr> <td>M8 (UNC 5/16")</td> <td>17, 106, 195, 284</td> </tr> </table>	g	L [mm]	M8 (UNC 5/16")	17, 106, 195, 284
g	L [mm]									
M8 (UNC 5/16")	24, 113, 202, 291									
g	L [mm]									
M8 (UNC 5/16")	17, 106, 195, 284									

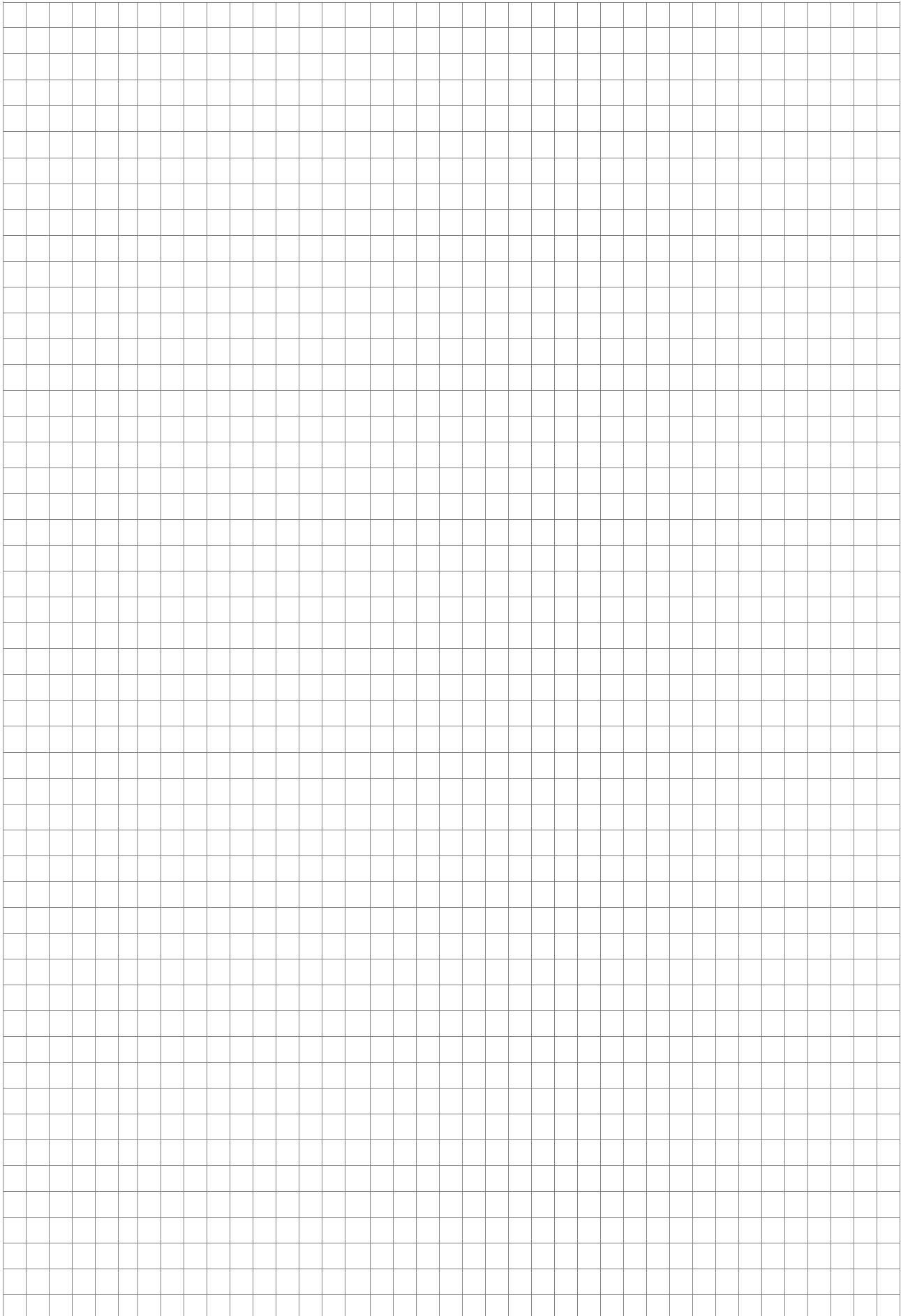
6.7.2 Temperature sensor

TSO	TSB
Oil temperature sensor (PT100)	Bearing temperature sensor (PT100)



1. Bearing temperature sensor position
2. Shock impulse sensor position
3. Oil temperature sensor position





Annex A

Lubrication unit



For a detailed description, please refer to the separate manual.

The designation of lubrication unit is set up as follows:

MHP29 HST3 FF25V L \ ..

Additional

Ex I = I Grade explosion-proof
Ex II = II Grade explosion-proof

Mounting type:

L = Left mounting
R = Right mounting

Filter module: $\beta_{25} \geq 75$

FF = Single filter
2FF = Duplex filter
V = Visual filter contamination indicator
E = Electrical filter contamination indicator

Cooler module:

HST = Oil/Water cooler
LAC = Oil/Air cooler
AH = Oil/Air cooler (Ex I)
LACX = Oil/Air cooler (Ex II)

Pump module

MHP = HP motor pump
MKF = KF Motor pump
SHP = Shaft end pump



If the pressure lubrication unit with duplex filter, only left mounting is available.

See selection tables for the list of lubrication unit modules.

Information to be given in order:

- Type of the unit
- Motor main voltage and frequency
- Instrument voltage and frequency
- Ambient temperature range
- Water temperature range (for oil/water coolers)
- Mounting position and location of the unit.

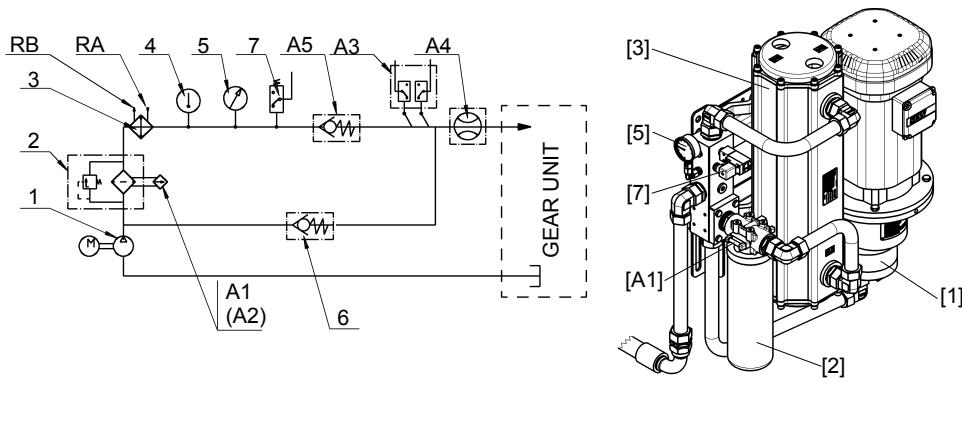
The mounting position is recommended to place the lubrication unit on the low speed shaft side. Other mounting position, please consult SEW-EURODRIVE.



Annex A

Lubrication unit

Flow diagram for lubrication unit with pipe cooler



Standard equipment:

- [1] Pump
- [2] Filter
- [3] Oil/Water cooler
- [5] Pressure gauge
- [6] Pressure relief valve
- [7] Pressure switch
- [A1] Visual filter contamination indicator
- [RA] Water inlet
- [RB] Water outlet

Optional equipment:

- [4] Thermometer
- [A2] Electrical filter contamination indicator
- [A3] Temperature monitors
- [A4] Flow indicator
- [A5] Check valve

Technical data for lubrication unit with oil/water cooler module (Frequency 50Hz)

Cooling rating P_c [kW]	Pump output Q_p [L/min]	Water supply Q_w (5-25°C) [L/min]	Water supply Q_w (25-40°C) [L/min]	Pump Module	Cooler Module	Motor Rating (50Hz 1500 rpm)		Water connections female		Unit weight [kg]
						IEC	[kW]	RA	RB	
5.3	18	8..12	12..25	MHP-12	HST1	24FF165	1.1	R3/4	R3/4	68
8.5	18	11..18	18..25	MHP-12	HST2	24FF165	1.1	R1	R1	70
10.5	28	14..25	25..40	MHP-18	HST2	24FF165	1.5	R1	R1	74
15	28	25..40	40..65	MHP-18	HST2	24FF165	1.5	R1	R1	74
19	44	35..45	45..85	MHP-29	HST3	28FF215	3	R1	R1	103
24	44	40..65	65..85	MHP-29	HST3	28FF215	3	R1	R1	103
25	65	35..60	60..80	MHP-47	HST3	28FF215	4	R1	R1	128
29	65	42..75	75..125	MHP-47	HST3	28FF215	4	R1	R1	128
36	81	50..75	75..130	MKF-63	HST4	38FF265	7.5	G1 1/2	G1 1/2	168
43	107	60..95	95..160	MKF-80	HST4	38FF265	7.5	G1 1/2	G1 1/2	167
48	107	75..100	100..175	MKF-80	HST5	38FF265	7.5	G2	G2	170
59	131	89..135	135..245	MKF-100	HST5	42FF300	11	G2	G2	222

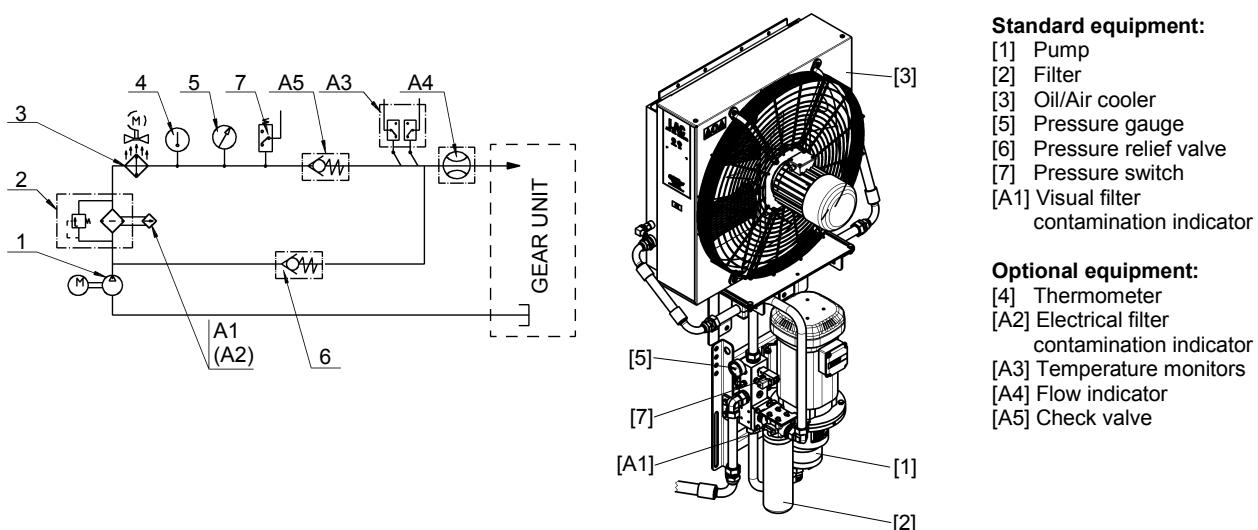
Technical data for lubrication unit with oil/water cooler module (Frequency 60Hz)

Cooling rating P_c [kW]	Pump output Q_p [L/min]	Water supply Q_w (5-25°C) [L/min]	Water supply Q_w (25-40°C) [L/min]	Pump Module	Cooler Module	Motor Rating (60Hz 1800 rpm)		Water connections female		Unit weight [kg]
						IEC	[kW]	RA	RB	
6.4	21.6	9..14	14..50	MHP-12	HST1	24FF165	1.5	R3/4	R3/4	73
9.5	21.6	15..35	35..80	MHP-12	HST2	24FF165	1.5	R1	R1	75
11	33.6	23..30	30..90	MHP-18	HST2	28FF215	2.2	R1	R1	85
16	33.6	30..65	65..90	MHP-18	HST2	28FF215	2.2	R1	R1	85
20	53	40..50	50..110	MHP-29	HST3	28FF215	3.7	R1	R1	105
26	53	45..80	80..110	MHP-29	HST3	28FF215	3.7	R1	R1	105
28	78	40..65	65..110	MHP-47	HST3	38FF265	5.5	R1	R1	147
32	78	45..92	92..130	MHP-47	HST3	38FF265	5.5	R1	R1	147
41	97	55..90	90..150	MKF-63	HST4	38FF265	9	R1 1/2	R1 1/2	205
46	128	65..100	100..170	MKF-80	HST4	38FF265	9	R1 1/2	R1 1/2	207
52	128	80..110	110..185	MKF-80	HST5	38FF265	9	R2	R2	211
63	157	95..145	145..255	MKF-100	HST5	42FF300	15	R2	R2	300

Annex A

Lubrication unit

Flow diagram for lubrication unit with air cooler



Technical data for lubrication unit with oil/air cooler module (Frequency 50Hz)

Cooling rating P _c [kW]	Pump output Q _p [L/min]	Pump Module	Cooler Module	Motor Rating (50Hz 1500rpm)		Fan Motor (50Hz)			Unit weight [kg]
				IEC	[kW]	[rpm]	[kW]	Noise level (1m) [dB (A)]	
5.3	18	MHP-12	LAC2 007-2-D	24FF165	1.1	3000	0.55	79	85
8.5	18	MHP-12	LAC2-011-2-D	24FF165	1.1	3000	1.1	82	95
10.5	28	MHP-18	LAC2 016-4-D	24FF165	1.5	1500	0.37	70	97
15	28	MHP-18	LAC2 023-4-D	24FF165	1.5	1500	0.75	76	109
19	44	MHP-29	LAC 033-6-A	28FF215	3	1000	0.55	74	144
24	44	MHP-29	LAC 033-4-A	28FF215	3	1500	2.2	84	151
25	65	MHP-47	LAC 033-4-A	28FF215	4	1500	2.2	84	175
29	65	MHP-47	LAC 044-4-A	28FF215	4	1500	2.2	85	188
36	81	MKF-63	LAC 056-6	38FF265	7.5	1000	1.5	81	217
43	107	MKF-80	LAC 056-6	38FF265	7.5	1000	1.5	81	228
48	107	MKF-80	LAC 056-4	38FF265	7.5	1500	2.2	84	228
59	131	MKF-100	LAC 078-6	42FF300	11	1000	2.2	87	345

Technical data for lubrication unit with oil/air cooler module (Frequency 60Hz)

Cooling rating P _c [kW]	Pump output Q _p [L/min]	Pump Module	Cooler Module	Motor Rating (60Hz 1800rpm)		Fan Motor (60Hz)			Unit weight [kg]
				IEC	[kW]	[rpm]	[kW]	Noise level (1m) [dB (A)]	
6.4	21.6	MHP-12	LAC2 007-2	24FF165	1.5	3600	0.66	86	90
9.5	21.6	MHP-12	LAC2-011-2	24FF165	1.5	3600	1.3	89	98
11	33.6	MHP-18	LAC2 016-4	28FF215	2.2	1800	1.3	89	109
16	33.6	MHP-18	LAC2 023-4	28FF215	2.2	1800	0.9	78	120
20	53	MHP-29	LAC 033-6	28FF215	3.7	1200	0.9	78	133
26	53	MHP-29	LAC 033-4	28FF215	3.7	1800	2.6	86	149
28	78	MHP-47	LAC 033-4	38FF265	5.5	1800	0.66	83	199
32	78	MHP-47	LAC 044-4	38FF265	5.5	1800	1.8	88	211
41	97	MKF-63	LAC 056-6	38FF265	9.2	1200	2.6	91	248
46	128	MKF-80	LAC 056-6	38FF265	9.2	1200	2.6	91	249
52	128	MKF-80	LAC 056-4	38FF265	9.2	1800	2.6	91	257
63	157	MKF-100	LAC 078-6	42FF300	15	1200	2.6	94	416



Maximum air temperature 35°C.

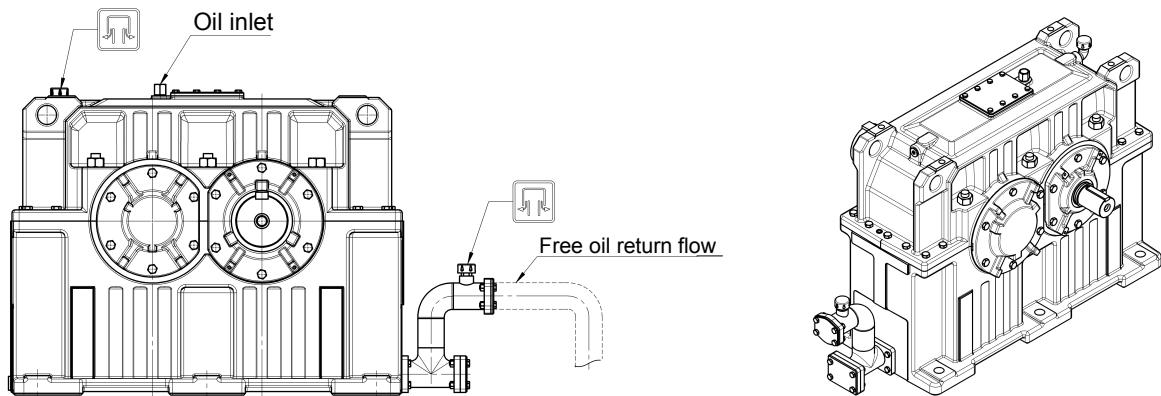


Annex B

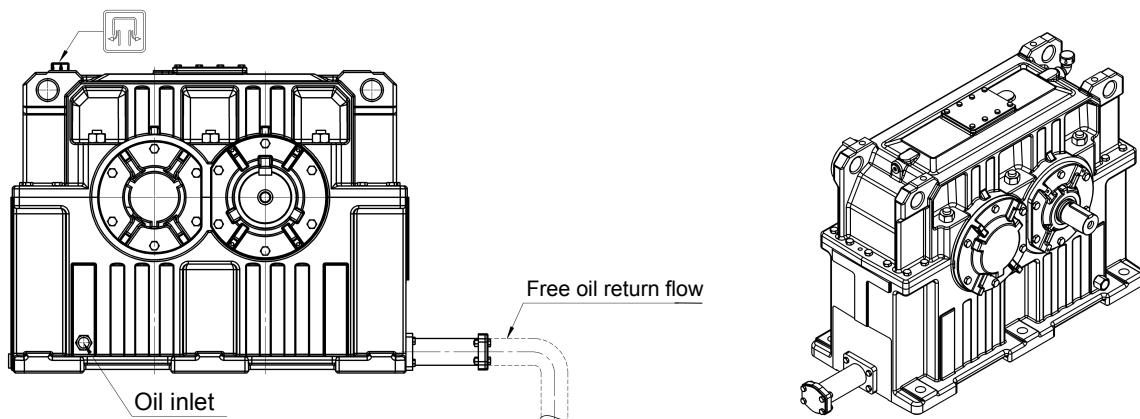
Central lubrication system connections

Equipment to connect the gear unit to the centralized oil filtering and cooling system.

Splash Lubrication



Pressure Lubrication



The diameter and height of the oil outlet pipe vary according to the gear unit size, lubrication method, oil viscosity and oil amount. SEW-EURODRIVE can deliver a visual or electrical flow meter to ensure the right oil flow into the gear unit.

Dimensions of the attachment flange depend on the pipe diameter and are according to the standard DIN 2642.

Annex C

Abbreviation key



f_1	Altitude factor (= Correction factor for calculating the thermal rating of the gear unit)	-
f_4	Operating cycle factor (= Correction factor for calculating the thermal rating of the gear unit)	-
f_T	Ambient temperature factor (= Correction factor for calculating the thermal rating of the gear unit)	-
f_L	Lubrication factor (= Correction factor for calculating the thermal rating of the gear unit)	-
F_A	Axial load	kN
F_F	Peak load factor	-
F_R	Radial load	kN
F_S	Service factor = $M_{N2} / M_{K2} = P_{N1} / P_{K1}$	-
$F_{S\ min}$	Application service factor	-
F_{start}	Startup factor	-
η	Efficiency	-
HSS	High speed shaft of the gear unit (usually input shaft)	-
i	Gear ratio	-
i_{ex}	Exact gear unit reduction ratio	-
i_N	Nominal gear unit (reduction) ratio	-
LSS	Low speed shaft of the gear unit (usually output shaft)	-
M_{K2}	Output torque (= operating torque on LSS)	kNm
$M_{K2\ max}$	Peak output torque (= peak operating torque on LSS)	kNm
$M_{K2\ ZUL}$	Permitted peak output torque	kNm
M_{N2}	Nominal gear unit torque	kNm
n_1	Input speed (HSS)	rpm
n_2	Output speed (LSS)	rpm
P_{K1}	Operating power on HSS	kW
$P_{K1\ max}$	Peak operating power on HSS	kW
$P_{K1\ ZUL}$	Permitted peak operating power on HSS	kW
P_{K2}	Operating power on LSS	kW
P_M	Nominal motor power	kW
P_{N1}	Nominal gear unit power (with reference to HSS)	kW
P_T	Thermal rating of the gear unit at existing ambient conditions	kW
P_{TH}	Thermal rating of the gear unit at defined ambient conditions	kW
P_L	Power loss to be cooled	kW



Annex D

Certificate



DET NORSKE VERITAS ACKNOWLEDGEMENT OF RECEIPT - EC

ACKNOWLEDGEMENT NO. 114660-2012-CE-RGC-DNV Rev.1

This Acknowledgement consists of 2 pages

This is to confirm that the Technical File for the following product(s):

Gear Units

with type designation(s)

See page 2

Manufactured by

SEW Industrial Gears (Tianjin) Co., Ltd.

No. 46, 7th Avenue, Tianjin Economic Technological Development Area,
Tianjin City, P.R.C

has been received and stored according to

the conformity assessment procedure described in Article 8.1.(b).(ii), of Council Directive 94/9/EC (ATEX) of 23 March 1994, category 2 non-electrical equipment.

Further details are given overleaf.

Place and date
Høvik, 2015-05-22
for DET NORSKE VERITAS AS

Ståle Sandstad

Ståle Sandstad
Certification Manager



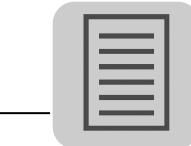
This Acknowledgement is valid until
2018-04-27

Notice: The acknowledgement is subject to terms and conditions overleaf. Any significant changes in design or construction may render this acknowledgement invalid.
The digitally signed and electronically distributed document is the original and valid acknowledgement. Ref.: www.dnv.com/digitalsignatures

If any person suffers loss or damage which is proved to have been caused by any negligent act or omission of Det Norske Veritas, then Det Norske Veritas shall pay compensation to such person for his proved direct loss or damage. However, the compensation shall not exceed an amount equal to ten times the fee charged for the service in question, provided that the maximum compensation shall never exceed USD 300,000. In this provision "Det Norske Veritas" shall mean the Foundation Det Norske Veritas as well as all its subsidiaries, directors, officers, employees, agents and any other acting on behalf of Det Norske Veritas.

Annex D

Certificate



Ack. No.: 114660-2012-CE-RGC-DNV,
Rev.1
Project No.: PRJC-377408-2012-PRC-CHN

Jurisdiction

DNV is appointed by the Directorate for Civil Protection and Emergency Planning as Notified Body (No. 0575) under the terms of the Norwegian regulation "Forskrift om utstyr og sikkerhetssystem til bruk i ekspløsjonsfarlig område", dated 1996-12-09 and Article 9 of Council Directive 94/9/EC (ATEX), as amended.

Acknowledgement history:

Revision	Description	Issue date
-	original Acknowledgement	2012-04-27
1.0	Renewal of acknowledgement	2015-05-22

Product description

The following types are covered by the Acknowledgement:

Product Description	Type Designations	Category	Product Group	Product Sub Group
Gear Units	M, M1,ML series	2	Ex equipment	Non-electrical

Technical documentation:

The following documentation has been received and stored:

Document No	Document Name
TCF- M&ML&M1 Series001, 04/2015	TCF-Industrial Gear Units of the M&ML&M1 Series

Terms and conditions

The product liability rests with the manufacturer, his representative or, in the absence of a representative, the importer, in accordance with the General Product Safety Directive 2001/95/EC. The following conditions may render this acknowledgement invalid:

- Changes in the design or construction of the product.
- Changes or amendments to the referenced directive(s).
- Changes or amendments in the standards which form the basis for documenting compliance with the essential requirements of the directive(s).

Conformity declaration and marking of product

In order to fully meet with the requirements of the Directive and legally affix the CE mark, the manufacturer must take all measures necessary to ensure that the manufactured product comply with the technical documentation and with the requirements of the Directive and finally draw up an EC declaration of conformity.

END OF ACKNOWLEDGEMENT



Annex D

Certificate



DNV BUSINESS ASSURANCE MANAGEMENT SYSTEM CERTIFICATE

Certificate No. 4039-1998-AQ-RGC-RvA

This is to certify that

SEW Industrial Gears (Tianjin) Co., Ltd.

No. 46, 7th Avenue, TEDA, Tianjin, P.R. China

has been found to conform to the Management System Standard:

ISO 9001:2008

This Certificate is valid for the following product or service ranges:

Sales, Design, Manufacture and Service of Gear Units and Segmented Girth Gears.

Initial Certification date:

December 11th, 1998

Place and date:

Shanghai, November 29th, 2013

This Certificate is valid until:
December 11th, 2016



for the Accredited Unit:
DNV CERTIFICATION B.V.,
THE NETHERLANDS

*The audit has been performed under the
supervision of*

Zhang Fu Wei
Lead Auditor


Chen Yi
Management Representative

Lack of fulfilment of conditions as set out in the Certification Agreement may render this Certificate invalid.

ACCREDITED UNIT: DET NORSKE VERITAS CERTIFICATION B.V., ZWOLSEWEIJK 1, 2994 LB, BARNEVELD, THE NETHERLANDS, TEL: +31 (0) 10 2922600 , www.DNVBA.COM

Annex D

Certificate



DNV·GL

MANAGEMENT SYSTEM CERTIFICATE

Certificate No:
127565-2012-AE-RGC-RvA

Initial certification date:
04 January, 2013

Valid:
28 December, 2015 - 14 September, 2018

This is to certify that the management system of

SEW Industrial Gears (Tianjin) Co., Ltd.

No. 46, 7th Avenue, TEDA, Tianjin, 300457, P.R. China

has been found to conform to the Environmental management system standard:

ISO 14001:2004

This certificate is valid for the following Scope:

**Sales, Design, Manufacture and Service of Gear Units and Segmented Girth
Gears**

Place and date:
Shanghai, 29 December, 2015



The RvA is a signatory to the IAF MLA

For the issuing office:
DNV GL – Business Assurance
Suite A, Building 9, No.1591
Hongqiao Road, Changning District,
Shanghai 200336, P.R. China
TEL: +86 21 32084518


Chen Yi
Management Representative

Lack of fulfilment of conditions as set out in the Certification Agreement may render this Certificate invalid.
ACCREDITED UNIT: DNV GL BUSINESS ASSURANCE B.V., ZWOLSEWEG 1, 2994 LB, BARENDRICHT, THE NETHERLANDS. TEL:+31102922689.
www.dnvba.com



Annex D

Certificate



DNV BUSINESS ASSURANCE

STATEMENT OF RECOGNITION

STATEMENT NO. 639 -15 - LAB30
The statement consists of 3 pages

This is to confirm that the

SEW Industrial Gears (Tianjin) Co., Ltd.

No.46, 7th Avenue, TEDA, Tianjin, China

has been found to comply with the requirements of DNV towards subcontractors of

Quality Management of Testing Laboratory

The acceptance is based on a formal Quality Audit and follow-ups according to relevant parts of ISO/IEC Guide 17025 (2005), in accordance with the requirements of the DNV Laboratory Quality Manual towards subcontractors.

Place and date

Shanghai, 30 September, 2015
for DNV GL Business Assurance
(China) Co., Ltd.

This Statement is valid until
29 September, 2018


Zhang Qing Ping
Product Compliance Manager

DNV local office:
DNV Shanghai


Sean Lee
Technical Reviewer

Notice: This Statement is subject to terms and conditions overleaf. Any significant change in the laboratory facilities or in the quality system may render this Statement invalid.
If any person suffers loss or damage which is proved to have been caused by any negligent act or omission of Det Norske Veritas, then Det Norske Veritas shall pay compensation to such person for his proved direct loss or damage. However, the compensation shall not exceed an amount equal to ten times the fee charged for the service in question, provided that the maximum compensation shall never exceed USD 2 million. In this provision "Det Norske Veritas" shall mean the Foundation Det Norske Veritas as well as all its subsidiaries, directors, officers, employees, agents and any other acting on behalf of Det Norske Veritas.

Annex D

Certificate



Statement No.: 639- 15- LAB30

Project No.: PRJC-506436-2014-PRC-CHN

Audit information

Initial audit:

- Date of Audit: 2015-06-11-12
- Initial Certification Audit Report: 2015-08-31/final
- Closing of Non-conformities: 2015-08-31

Scope of recognition

JB/T 8555 2008
GB/T 7232 2012
GB/T 9452 2012
GB/T 2828.1 2012
JB/T 6050 2006
GB/T 230.1 2009
GB/T 231.1 2009
GB/T 4340.1 2009
GB/T 4342 1991
GB/T 1818 1994
GB/T 13298 1991
GB/T 13299 1991
GB/T 2975 1998
GB/T 228.1 2010
GB/T 229 2007
GB/T 232 2010
GB/T 225 2006
GB/T 8539 2000
ISO 6336-5 2003
DIN EN 10083-1 2006
DIN EN 10083-2 2006
DIN EN 10083-3 2006
DIN EN 10084 2008
GB/T 3077 1999
GB/T 222 2006
GB/T 14203 1993
GB/T 4336 2002
SEW-SAT 1900E 6.3.2012
GB/T 5617 2005
JB/T 9204 2008
GB/T 9450 2005
GB/T 9451 2005
QC/T 262 1999
JB/T 3999 2007
GB/T 18177 2008
GB 11354 1989
GB/T 13320 2007



Annex D

Certificate



Statement No.: 639- 15- LAB30

Project No.: PRJC-506436-2014-PRC-CHN

GB/T 10561 2005
GB/T 1979 2001
GB/T 6394 2002
GB/T 226 1991
GB/T 3098.1 2010

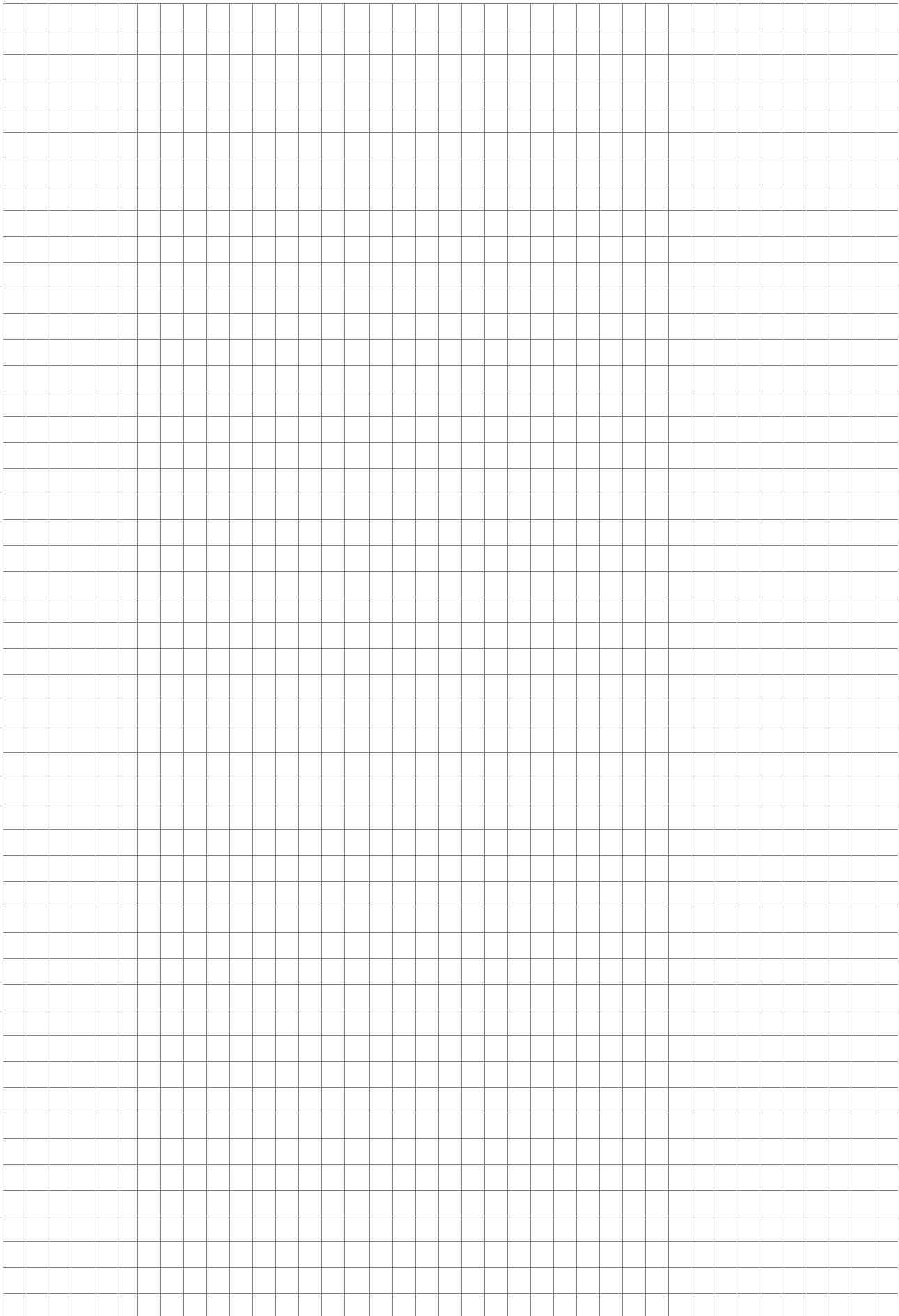
Application/Limitations

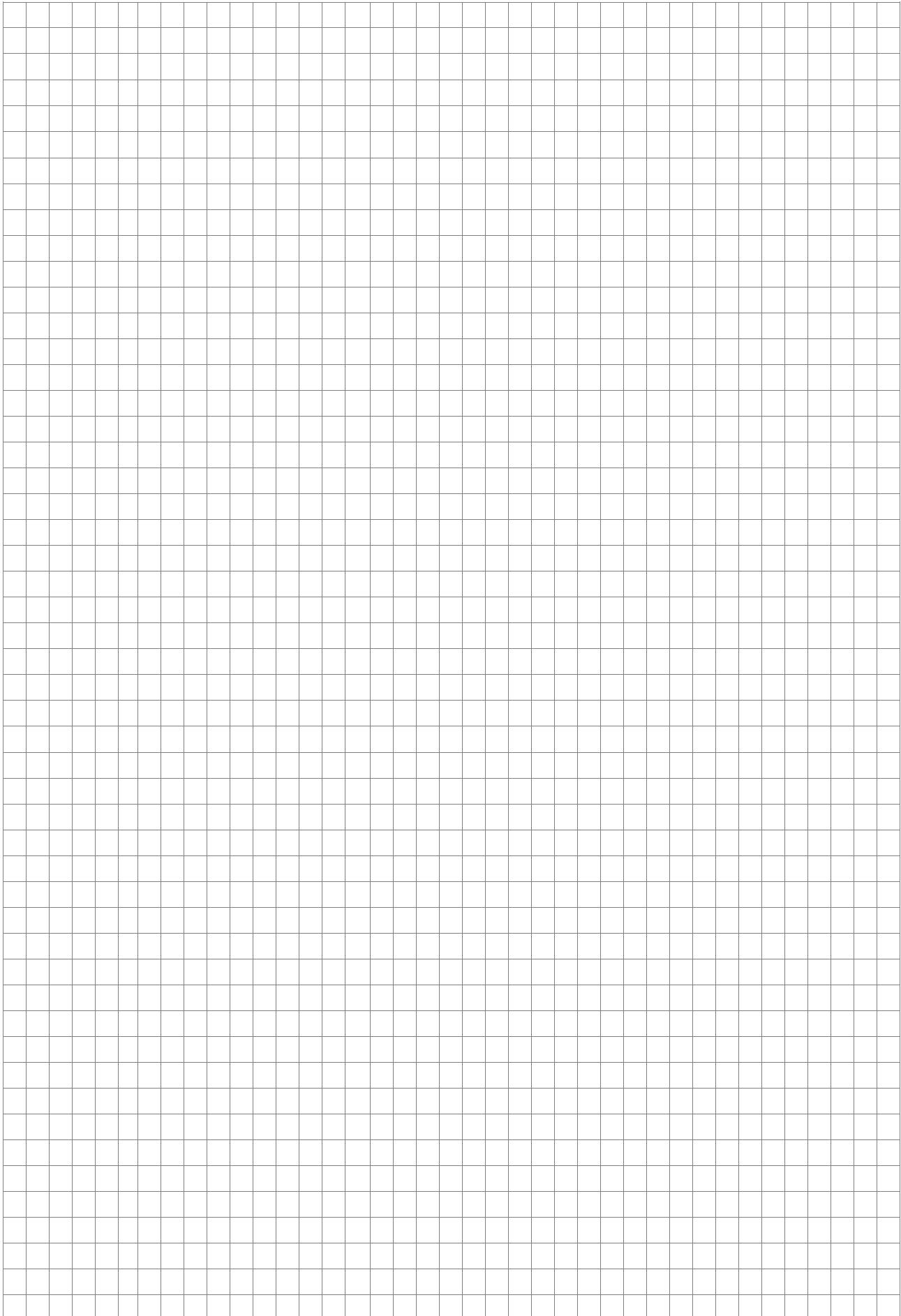
Related light/heavy industry field.

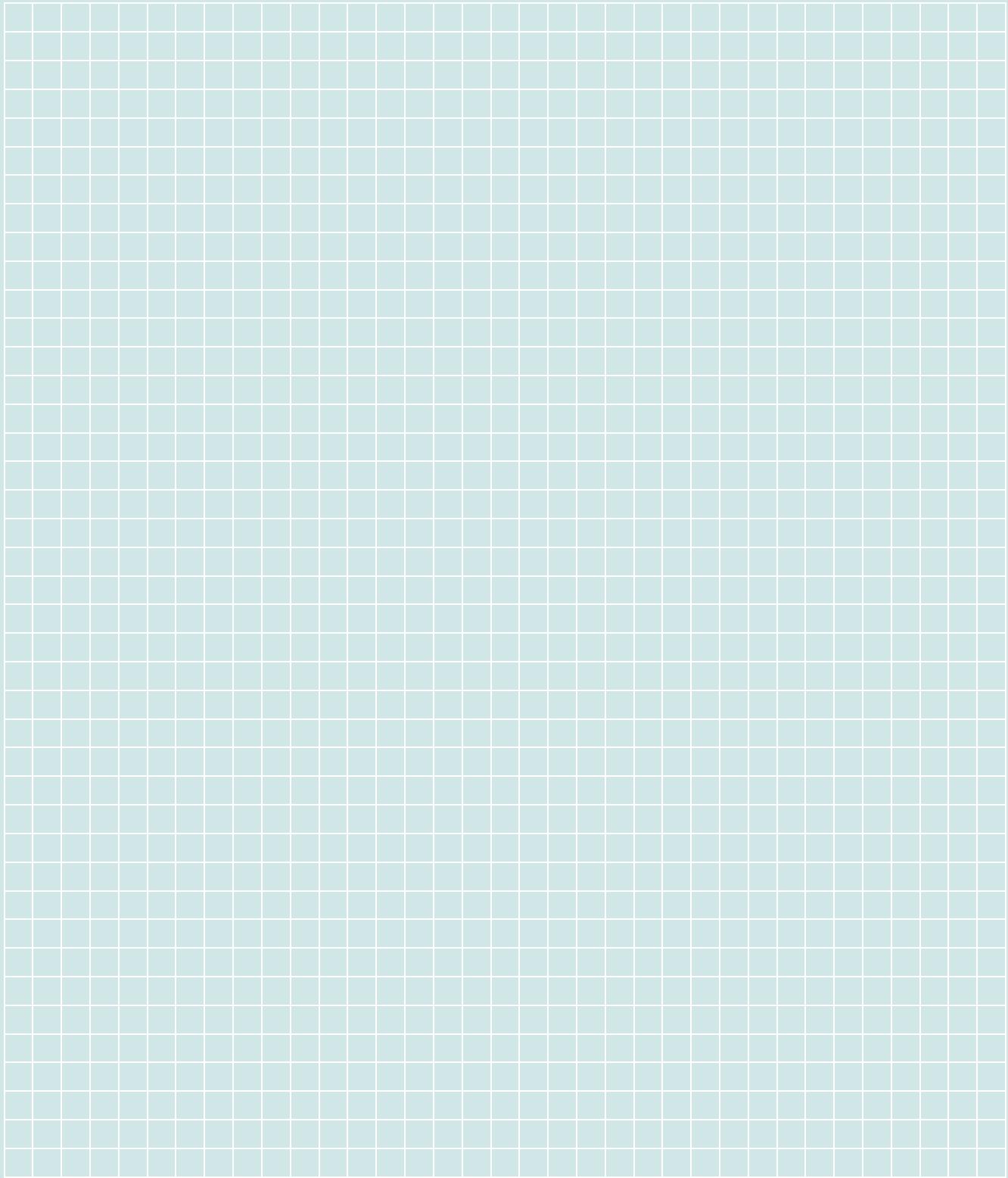
Documentation

LAB/SIG-QM for the QM-quality manual; LAB/SIG-xx-xx for procedure and for daily operation working instruction as well.

END OF STATEMENT









SEW-EURODRIVE
Driving the world

SEW
EURODRIVE